

Beschlussempfehlung und Bericht

des Ausschusses für Verkehr, Bau- und Wohnungswesen (15. Ausschuss)

- 1. zu dem Antrag der Abgeordneten Wolfgang Börnsen (Bönstrup), Dirk Fischer (Hamburg), Eduard Oswald, weiterer Abgeordneter und der Fraktion der CDU/CSU**
– Drucksache 14/6316 –

Frontpartien von Fahrzeugen europaweit fußgängersicher gestalten

- 2. zu der Unterrichtung durch die Bundesregierung**
– Drucksache 14/7409 Nr. 2.1 –

Mitteilung der Kommission an den Rat und das Europäische Parlament
Fußgängerschutz:
Selbstverpflichtung der europäischen Automobilindustrie
KOM (2001) 389 endg.; Ratsdok. 09616/01

A. Problem

Die Antragsteller haben einen Antrag mit dem Ziel gestellt, der Bundestag solle die Bundesregierung auffordern, im Rahmen der Europäischen Union auf eine zügige Einführung von technischen Karosserie- und Karosseriezubehörvorschriften bezüglich der Frontpartien von Kraftfahrzeugen zum Schutz von Passanten und Fahrradfahrern bei Unfällen hinzuwirken und kurzfristig bis zur Einführung entsprechender europäischer Vorschriften auf nationaler Ebene eine diesbezügliche Änderung der Straßenverkehrszulassungsverordnung durchzuführen.

B. Lösung

Erledigterklärung des Antrags auf Drucksache 14/6316 und einstimmige Annahme einer Entschließung

C. Alternativen

Keine

D. Kosten

Wurden nicht erörtert.

Beschlussempfehlung

Der Bundestag wolle beschließen,

1. den Antrag auf Drucksache 14/6316 für erledigt zu erklären und
2. stattdessen sowie zu der Unterrichtung durch die Bundesregierung – Drucksache 14/7409 Nr. 2.1 – in Kenntnis dieser Unterrichtung folgende Entschließung anzunehmen:
 - a. Der Deutsche Bundestag unterstützt das Vorgehen der Bundesregierung im Rahmen der Europäischen Union auf eine zügige Einführung von technischen Karosserie- und Karosseriezubehörvorschriften bezüglich der Frontpartien von Kraftfahrzeugen zum Schutz von Passanten und Fahrradfahrern bei Unfällen hinzuwirken;
 - b. der Deutsche Bundestag unterstützt die Bundesregierung in ihrem Bemühen, auf EU-Ebene insbesondere ein rasches Verbot von Frontbügeln an Kraftfahrzeugen zu erreichen. Um diesem Anliegen Nachdruck zu verleihen, wird die Bundesregierung gebeten, auf nationaler Ebene eine diesbezügliche Änderung der Straßenverkehrszulassungsverordnung durchzuführen und über die Ergebnisse den Deutschen Bundestag zu informieren.

Berlin, den 13. März 2002

Der Ausschuss für Verkehr, Bau- und Wohnungswesen

Eduard Oswald
Vorsitzender

Hans-Günter Bruckmann
Berichtersteller

Bericht des Abgeordneten Hans-Günter Bruckmann

I. Überweisung

Der Deutsche Bundestag hat den Antrag auf Drucksache 14/6316 in seiner 182. Sitzung am 5. Juli 2001 beraten und an den Ausschuss für Verkehr, Bau- und Wohnungswesen zur federführenden Beratung und an den Rechtsausschuss sowie den Ausschuss für Wirtschaft und Technologie zur Mitberatung überwiesen.

II. Wesentlicher Inhalt der Vorlagen

Die Antragsteller haben einen Antrag mit dem Ziel gestellt, der Bundestag solle die Bundesregierung auffordern, im Rahmen der Europäischen Union auf eine zügige Einführung von technischen Karosserie- und Karosseriezubehörvorschriften bezüglich der Frontpartien von Kraftfahrzeugen zum Schutz von Passanten und Fahrradfahrern bei Unfällen hinzuwirken und kurzfristig bis zur Einführung entsprechender europäischer Vorschriften auf nationaler Ebene eine diesbezügliche Änderung der Straßenverkehrszulassungsverordnung durchzuführen.

III. Stellungnahme der mitberatenden Ausschüsse

Der **Rechtsausschuss** hat die Vorlage in seiner 119. Sitzung am 13. März 2002 beraten und empfiehlt einstimmig, den Antrag in der Fassung der Ausschussdrucksache 992 des federführenden Ausschusses (diese entspricht inhaltlich den Teilen a und b der Beschlussempfehlung) anzunehmen.

Der **Ausschuss für Wirtschaft und Technologie** hat die Vorlage in seiner 76. Sitzung am 13. März 2002 beraten. Er hat mitgeteilt, er unterstütze das Bemühen des federführenden Ausschusses, bei der Thematik der fußgängersicheren Gestaltung der Frontpartien von Fahrzeugen zu einem interfraktionellen Antrag zu kommen und habe daher auf eine Abstimmung des Antrages der Fraktion der CDU/CSU verzichtet.

IV. Beratungsverlauf und Beratungsergebnisse im Ausschuss

Der Ausschuss für Verkehr, Bau- und Wohnungswesen hat den Antrag in seiner 79. Sitzung am 13. März 2002 beraten.

Während der Sitzung zeichnete sich die Verständigung auf einen gemeinsamen Antrag aller Fraktionen ab. Der Berichterstatter und die Bundesregierung machten Formulierungsvorschläge. Die Beratung der Vorlage wurde daher unterbrochen, um die Möglichkeit zu geben, sich am Rande der Sitzung auf die Formulierung einer gemeinsamen Entschließung – auch zu der EU-Vorlage Ratsdok. 09616/01 – zu einigen. Nach der Verständigung auf eine gemeinsame Entschließung aller Fraktionen (deren Text sich aus Teil a und Teil b der Beschlussempfehlung ergibt), wurde die Beratung der beiden Vorlagen wieder aufgenommen. Die Fraktion der FDP hatte im Ausschuss einen Antrag eingebracht, der sich aber aufgrund der Einigung über einen gemeinsamen Antrag aller Fraktionen erledigte. Der Ausschuss empfiehlt einstimmig, den Antrag der Fraktion der CDU/CSU auf Drucksache 14/6316 für erledigt zu erklären und stattdessen sowie zu der Unterrichtung durch die Bundesregierung den Entschließungsantrag aller Fraktionen zu dem Antrag der Fraktion der CDU/CSU auf Drucksache 14/6316 und zu der Unterrichtung durch die Bundesregierung – Drucksache 14/7409 Nr. 2.1 – anzunehmen.

Die **Fraktion der SPD** erklärte, sie sei der Auffassung, dass der Antrag der CDU/CSU-Fraktion durch Regierungshandeln erledigt sei, weil die Bundesregierung auf EU-Ebene auf eine europaweit geltende Selbstverpflichtung der Automobilindustrie hinwirke. Es sei aber das gemeinsame Ziel, für die Abschaffung der Frontschutzbügel einzutreten.

Die **Fraktion der CDU/CSU** führte aus, sie sei der Auffassung, dass – solange es keine europaeinheitliche Lösung gebe – eine nationale Lösung erforderlich sei. Bei einer vernünftigen, vertretbaren und fairen Formulierung, die auch Rücksicht auf die Initiatoren nehme, werde man sich einem gemeinsamen Vorgehen nicht verschließen.

Die **Fraktion BÜNDNIS 90/DIE GRÜNEN** schloss sich den Ausführungen der Fraktion der SPD an.

Die **Fraktion der FDP** erklärte, sie unterstütze eine gemeinsame Lösung des Problems.

Die **Fraktion der PDS** führte aus, dass der Antragsgegenstand eine längst überfällige Diskussion betreffe. Frontschutzbügel gebe es seit über zwanzig Jahren. Einer gemeinsamen Lösung werde man sich anschließen.

Berlin, den 13. März 2002

Hans-Günter Bruckmann
Berichterstatter



KOMMISSION DER EUROPÄISCHEN GEMEINSCHAFTEN

Brüssel, den 11.7.2001
KOM(2001) 389 endgültig

**MITTEILUNG DER KOMMISSION
AN DEN RAT UND DAS EUROPÄISCHE PARLAMENT**

**Fußgängerschutz:
Selbstverpflichtung der europäischen Automobilindustrie**

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**Fußgängerschutz:
Selbstverpflichtung der europäischen Automobilindustrie**

1. Einleitung

Am 21. Dezember 2000 erörterte die Kommission die Möglichkeiten einer Selbstverpflichtung seitens der Industrie zur Verbesserung des Schutzes von Fußgängern und anderen Verkehrsteilnehmern vor Verletzungen beim Zusammenprall mit Kraftfahrzeugen.

Am 21. Dezember 2000 verabschiedete die Kommission ferner eine Mitteilung, in der sie beschloss, Gespräche mit dem Dachverband der Europäischen Automobilhersteller ACEA aufzunehmen und die Industrie zu einer Selbstverpflichtung zu bewegen, die den Zielvorstellungen der Kommission auf diesem Gebiet gerecht wird. Die Kommission befürwortete darüber hinaus parallele Verhandlungen mit den Dachverbänden der japanischen und koreanischen Fahrzeughersteller JAMA bzw. KAMA. Zusätzlich beschloss die Kommission, die Arbeiten an einem Richtlinienvorschlag parallel dazu weiterzuführen, um für den Fall gerüstet zu sein, dass die Industrie keine zufriedenstellende Selbstverpflichtung anbieten würde. Die Entscheidung, ob die Selbstverpflichtung der Industrie angenommen werden oder eine Richtlinie vorgeschlagen werden sollte, war für Mitte 2001 geplant.

Am 6. Februar 2001 fand eine öffentliche Anhörung statt, bei der alle Interessengruppen, auch die Mitgliedstaaten und Vertreter des Europäischen Parlaments, ihre Ansichten über Vorschriften zur Verbesserung des Fußgängerschutzes darlegen konnten; die einen befürworteten eine ausgehandelte Selbstverpflichtung, die anderen sprachen sich für eine Rechtsvorschrift aus. Die Frage des Fußgängerschutzes wurde darüber hinaus in der Arbeitsgruppe ‚Kraftfahrzeuge‘ der Kommission erörtert (*Motor Vehicle Working Group - MVWG*), der auch Interessengruppen angehören.

Inzwischen sind die Verhandlungen mit der europäischen Automobilindustrie erfolgreich abgeschlossen. Wenn auch die parallelen Verhandlungen mit der japanischen und koreanischen Automobilindustrie beendet sind, will die Kommission zunächst das Europäische Parlament und den Rat informieren und danach eine Empfehlung verabschieden, die an diese Industrieverbände gerichtet ist.

Die vorliegende Mitteilung erläutert, wie die Kommission die Bedingungen der Selbstverpflichtung einschätzt.

2. Selbstverpflichtung der Industrie

Nach der Aufforderung durch die Kommission in der Mitteilung vom 21. Dezember 2000 führten die Kommissionsdienststellen Gespräche mit ACEA; die Gespräche mit JAMA und KAMA sind noch nicht abgeschlossen.

Die Selbstverpflichtung von ACEA (siehe Anlage zu dieser Mitteilung) orientiert sich an der bestehenden Selbstverpflichtung von ACEA, JAMA und KAMA zur Verringerung des CO₂-Ausstoßes von Personenkraftwagen und enthält folgende Elemente:

- 1) Die Industrie verpflichtet sich in einem ersten Schritt, die Empfehlungen der Gemeinsamen Forschungsstelle der Kommission GFS zur Verbesserung der passiven Sicherheit umzusetzen: Ab 1. Juli 2005 sollen alle neuen Fahrzeugtypen die Anforderungen erfüllen, ab dem 1. Juli 2010 dann 80 % aller Neufahrzeuge, im Jahr 2011 90 % und bis 2012 die restlichen 10 %.
- 2) Die Industrie verpflichtet sich, im Jahr 2003 alle Neufahrzeuge mit Antiblockierbremsystemen (ABS) und im Jahr 2002 mit Tagesfahrlicht auszurüsten.
- 3) Starre Frontschutzbügel werden ab 2002 bei Neufahrzeugen nicht mehr installiert.
- 4) Die Industrie unterstützt die Zielsetzung des Aktionsplans *e-Europe* und hält zusätzliche IKT-Systeme zur Verbesserung der aktiven Sicherheit für erforderlich (eine vorläufige Liste findet sich im Anhang zur Selbstverpflichtung). Die Industrie verpflichtet sich ferner, nach und nach alle neuen Kraftfahrzeuge mit zusätzlichen aktiven Sicherheitseinrichtungen auszustatten.
- 5) Die Industrie verpflichtet sich, 2010 bei allen neuen Fahrzeugtypen und ab 2012 (bis spätestens Ende 2014) nach und nach bei allen Neufahrzeugen die Zielvorgaben des Europäischen Ausschusses für die Verbesserung der Fahrzeugsicherheit (*European Enhanced Safety Vehicle Committee - EEVC*) auf dem Gebiet des Fußgängerschutzes einzuhalten; die Grundlage dafür bilden technische Vorschriften der EEVC oder sonstige Maßnahmen, die diesen zumindest ebenbürtig sind (zumindest gleiche Schutzwirkung), allerdings vorbehaltlich der Beurteilung ihrer Durchführbarkeit. Für diese Beurteilung zum 1. Juli 2004 ist ein Begleitausschuss zuständig, den die Kommission mit Beteiligung der Industrie einsetzen wird. Die Beurteilung wird sich auf Erkenntnisse unabhängiger Gremien stützen und auch die Umsetzungsfristen für die EEVC-Anforderungen bei Fahrzeugen berücksichtigen, die die Fristen in Ausnahmefällen nicht einhalten können.

Die Vollzugsmeldungen sowie die Berichte über technische Fortschritte und Planungsfragen werden an den Begleitausschuss gerichtet. Die Überprüfung der technischen Konformität wird von unabhängigen technischen Stellen durchgeführt.

Darüber hinaus enthält die Selbstverpflichtung eine Reihe allgemeiner Bestimmungen; besonders hervorzuheben sind folgende Punkte:

- Internationale Harmonisierung: Die Automobilindustrie unterstützt künftige Bemühungen zur internationalen Harmonisierung auf dem Gebiet des Fußgängerschutzes im Rahmen des UN/ECE-Übereinkommens aus dem Jahr 1998 über die Einführung globaler technischer Regelungen.
- Konformität mit EG-Wettbewerbsvorschriften: Die Selbstverpflichtung wird im Einklang mit den Wettbewerbsvorschriften der EG umgesetzt. Falls die Selbstverpflichtung bestimmte Wettbewerbsbeschränkungen beinhaltet, könnte der Kommission eine förmliche Mitteilung zugestellt werden, sofern geltende EG-Wettbewerbsvorschriften dies vorsehen.

3. Empfehlung der Kommission

Als zusätzliche Sicherheitsmaßnahme dafür, dass ACEA die Selbstverpflichtung auch einhält, wird die Kommission in ihrer Empfehlung klar stellen, dass sie ordnungspolitische Maßnahmen erwägen wird, falls ACEA seiner Selbstverpflichtung nicht nachkommt.

4. Beurteilung der Selbstverpflichtung

Im Dezember 2000 verständigte sich die Kommission darauf, dass eine Selbstverpflichtung der Industrie zur Fußgängersicherheit folgende Elemente umfassen muss:

- Sie muss die Verkehrssicherheit verbessern; insbesondere muss sie Lösungen anbieten, die die Anforderungen der EEVC weitgehend erfüllen.
- Sie muss bis spätestens Juni 2001 zum Abschluss kommen.
- Sie muss angemessene Kontrollmöglichkeiten bieten, damit überwacht werden kann, ob die Industrie ihre Zusagen einhält.
- Sie muss die Bestimmungen der Mitteilung über Umweltvereinbarungen aus dem Jahr 1996 berücksichtigen.
- Sie muss einen angemessenen Teil der Automobilindustrie abdecken.

Die Kommission hat die Selbstverpflichtung von ACEA im Lichte der genannten Elemente begutachtet und ist zu folgendem Schluss gelangt:

1) Vorteile für die Verkehrssicherheit

ACEA ist bereit, einen Fußgängerschutz auf hohem Niveau zu gewährleisten; dies soll durch Einhaltung der EEVC-Anforderungen geschehen oder durch Maßnahmen, die 2010 ein zumindest gleichwertiges Schutzniveau bieten. Als erster Schritt in diese Richtung hat sich die Industrie bereit erklärt, ab 1. Juli 2005 die von der GFS in ihrem Bericht vom 19. Dezember 2000 vorgeschlagenen Anforderungen zu erfüllen. Im Übrigen ist die Industrie zu drei zusätzlichen Initiativen bereit, die ebenfalls zur Verbesserung des Fußgängerschutzes und der Straßenverkehrssicherheit beitragen, nämlich die Ausrüstung aller neuen Kraftfahrzeuge mit Antiblockierbremsystemen (ABS) im Jahr 2003 und mit Tagesfahrlicht im Jahr 2002 sowie die Einstellung des Vertriebs von starren Frontschutzbügeln durch die Fahrzeughersteller ab 2002.

2) Zeitpunkt des Inkrafttretens

Laut Selbstverpflichtung wird die Industrie das erste bedeutende Maßnahmenpaket zur Verbesserung der Fußgängersicherheit bis zum 1. Juli 2005 umsetzen, aber schon im Zeitraum 2002-2004 wird sie wichtige Zusatzmaßnahmen ergreifen (siehe vorangehenden Absatz).

3) Einhaltung der Bestimmungen

Die Industrie ist damit einverstanden, dass jährlich ermittelt wird, welche Fortschritte mit der Selbstverpflichtung erzielt wurden (auch auf dem Gebiet der Forschung und Entwicklung) und in welchem Maße die Anforderungen erfüllt wurden. Für die Überwachung wird ein Begleitausschuss zuständig sein, der von der Kommission mit Beteiligung der Industrie

eingesetzt werden soll. Die Überwachung erfolgt auf der Grundlage der Prüfungen, die von unabhängigen technischen Stellen durchgeführt werden.

Bei der praktischen Umsetzung des Überwachungsverfahrens wird sich die Kommission von folgenden Grundsätzen leiten lassen:

Der Begleitausschuss wird von der Kommission verwaltet.

Die Kommission wird bei Bedarf Sachverständige in den Begleitausschuss berufen. Sie sollen bei der technischen Begutachtung der Berichte helfen, die von der Industrie vorgelegt werden.

Falls kein Konsens erreicht wird, kann die Kommission die Gutachten ihrer Dienststellen als Maßstab für ihren Bericht an das Europäische Parlament und den Rat verwenden.

4) Abstimmung mit der Mitteilung über Umweltvereinbarungen aus dem Jahr 1996

Die Selbstverpflichtung erfüllt die in der Mitteilung aus dem Jahr 1996 aufgestellten Kriterien; dies gilt besonders für folgende Punkte:

- Anhörung: Die Interessengruppen wurden konsultiert; dazu dienten die Anhörung vom 6. Februar 2001 sowie Informationssitzungen mit den zuständigen Ministerien der Mitgliedstaaten, einigen Mitgliedern des Europäischen Parlaments und Nichtregierungsorganisationen in Schlüsselbereichen (Europäischer Rat für Verkehrssicherheit). Die förmliche Konsultation des Europäischen Parlaments und des Rates erfolgt mit der vorliegenden Mitteilung.
- Quantifizierte Ziele: Die Selbstverpflichtung enthält quantitativen Zielvorgaben für die technischen Anforderungen und die Prüfung.
- Überwachung der Ergebnisse: Die Erfüllung der Zielvorgaben wird jährlich geprüft (siehe vorangehenden Abschnitt).
- Unabhängige Prüfung der Ergebnisse: Die Erfüllung der Zielvorgaben wird durch Dritte überprüft (siehe vorangehenden Abschnitt).
- Information der Öffentlichkeit und Transparenz: Die Selbstverpflichtung wird als Anhang zur vorliegenden Mitteilung veröffentlicht.

Die vorliegende Mitteilung erfolgt unbeschadet der allgemeinen Rahmenbedingungen für freiwillige Vereinbarungen, die die Kommission möglicherweise noch verabschiedet wird.

5) Abdeckung des Marktes

Die Selbstverpflichtung von ACEA deckt über 80 % des Gemeinschaftsmarktes für Personenkraftwagen und leichte Nutzfahrzeuge ab. Sobald JAMA und KAMA parallele Selbstverpflichtungen unterzeichnet haben, werden praktisch alle in der EG abgesetzten Personenkraftwagen und leichten Nutzfahrzeuge unter 2,5 Tonnen davon betroffen sein.

5. Weitere Schritte

Falls die Kommission die Selbstverpflichtung der ACEA akzeptiert, müssen in den darauffolgenden 18 Monaten zwei wichtige Schritte unternommen werden:

- Die Kommission muss den Begleitausschuss einsetzen, zu dem auch Vertreter der Automobilindustrie eingeladen werden, und die Zusammensetzung und die Geschäftsordnung dieses Ausschusses festlegen.
- Gemeinsam mit der Industrie müssen unabhängige technische Stellen ermittelt werden, die in der Lage sind, die in der Selbstverpflichtung dargelegten Prüfungen durchzuführen.

Die Kommission beabsichtigt, das Europäische Parlament und den Rat regelmäßig über die Umsetzung der Selbstverpflichtung zu informieren.

6. Schlussfolgerungen

Nach Ansicht der Kommission entspricht die Selbstverpflichtung von ACEA dem in der Mitteilung vom 21. Dezember 2000 erteilten Mandat. Die Selbstverpflichtung wäre ein entscheidender und progressiver Beitrag zur Erreichung der vorrangigen Gemeinschaftsziele auf dem Gebiet der Straßenverkehrssicherheit in der EU.

Bevor die Kommission jedoch darüber befindet, ob sie die Selbstverpflichtung akzeptieren will, möchte sie mit der vorliegenden Mitteilung das Europäische Parlament und den Rat vom Inhalt der Selbstverpflichtung von ACEA und der Einschätzung der Kommission in Kenntnis setzen. Aus diesem Grund wird die Kommission bis spätestens Dezember 2001 abwarten. Dann wird sie darüber entscheiden, ob sie die Selbstverpflichtung mittels einer Empfehlung an die Industrie akzeptiert oder eine Richtlinie auf der inhaltlichen Grundlage der Selbstverpflichtung vorschlägt, die entweder als Rahmenrichtlinie oder als Richtlinie herkömmlicher Art ausgelegt sein könnte. In der Zwischenzeit werden die Arbeiten an dem Entwurf einer Rechtsvorschrift weitergeführt.

ANHANG

Selbstverpflichtung von ACEA zum Schutz von Fußgängern und Radfahrern

Der Anhang liegt nur in Englisch vor.

ACEA COMMITMENT

RELATING TO THE PROTECTION OF PEDESTRIANS AND CYCLISTS

The Commitment has to be considered null and void if MG Rover¹ and the other car manufacturers selling into the EU market represented by JAMA and KAMA have not undertaken an equivalent Commitment by the date of adoption of the recommendation by the Commission.

INTRODUCTION

1. The European Automobile Manufacturers Association (hereinafter “ACEA”) and its members recognise the need to improve the protection of pedestrians and other road users from injury stemming from a collision with motor vehicles, which was identified as a key priority in EU road safety in the Communication of the European Commission of 17 March 2000² and taking also into consideration the subsequent Resolution of the Council of 26 June 2000 and the Resolution of the European Parliament of 18 January 2001.
2. This Commitment reflects ACEA’s support of the Community’s priorities on pedestrian protection, while at the same time it allows a more rapid introduction of a comprehensive package of active and passive safety measures in this area than would otherwise be the case through legislation, as well as a more flexible response to new research findings and new technologies.
3. This Commitment also reflects the conclusions of the Lisbon Summit in which the European Council asked the Commission to “*set out by 2001 a strategy for further co-ordinated action to simplify the regulatory environment.*” The Council also stated that “*the speed of technological change may require new and flexible regulatory approaches.*”

In the context of the ongoing debate on new forms of European governance, it is appropriate to look at new forms of policy co-operation between the public and private sectors to complete the Single Market and to stimulate European competitiveness.

Furthermore, ACEA supports Community efforts to introduce this Commitment as a candidate item for international harmonisation within the UN/ECE Global Agreement.

PARTIES

4. This Commitment is undertaken by ACEA with the support of its car manufacturing member companies (BMW, Fiat, Ford Group, GM, DaimlerChrysler AG, Porsche, PSA Peugeot Citroën, Renault and Volkswagen), for motor vehicles to which this Commitment applies whether manufactured in the EU or outside, and imported into the EU.
5. ACEA and its members will make every effort to ensure that other motor vehicle manufacturers selling in the EC the categories of vehicles to which this Commitment applies, and in which ACEA members or their parent companies have sole or joint control, also commit themselves to the provisions of this Commitment.
6. ACEA will ensure that the Commitment is implemented in a manner which complies with applicable EC competition rules.

¹ Contacts have been made by ACEA with MG Rover which should resolve its participation.

² COM(2000) 125 final.

SCOPE

7. This Commitment applies to all motor vehicles of category M1, of a total permissible mass not exceeding 2.5 tonnes, and to N1 vehicles derived from M1, of a total permissible mass not exceeding 2.5 tonnes, as defined in Directive 70/156/EEC³, as last amended by Directive 98/91/EC⁴.

ACEA COMMITMENTS

First phase

8. From 1 July 2005 as target date, but no later than 1 October 2005, all new types of motor vehicles to which this Commitment applies placed on the market by ACEA members will comply with the provisions set forth in this Commitment and part A of annexes II and III.
9. From 1 July 2010 80% and in 2011 90% of all new motor vehicles to which this Commitment applies placed on the market by ACEA members will comply with the provisions set forth in this Commitment and part A of annexes II and III, and the remaining 10% by 31 December 2012 at the latest.

Second phase

10. Not later than in 2010 all new types of motor vehicles to which this Commitment applies placed on the market by ACEA members will comply with the provisions set forth in this Commitment and with the targets set forth in part B of annex II through technical prescriptions in part B of annex III or other measures which are at least equivalent (at least equal protective effects), subject to a feasibility assessment. This feasibility assessment will be undertaken in the frame of the Monitoring Committee referred to in paragraph 17, taking into consideration the findings of independent bodies, by 1 July 2004.
11. Progressively from 2012, but not later than 5 years after the beginning of phase B, all new motor vehicles to which this Commitment applies placed on the market by ACEA members will comply with the provisions set forth in this Commitment and with the targets set forth in part B of annex II through technical prescriptions (part B annex III) or other measures which are at least equivalent (at least equal protective effects).

The assessment referred to in paragraph 10 will also cover the date from which those vehicles which in exceptional cases are unable to meet the deadline referred to in the previous sub-paragraph will comply with the provisions mentioned in it.

12. ACEA and its members will undertake further research in the areas covered by paragraphs 8 and 10. Joint efforts will be considered within current and future Framework programmes.

Active safety and ICT elements

13. During 2003, at least 90% of all new motor vehicles, and from 1 July 2004 all new motor vehicles to which this Commitment applies placed on the market by ACEA members will be equipped with Anti-lock Brake Systems (ABS).
14. During 2002, at least 90% of all new motor vehicles and from 1 October 2003 all new motor vehicles to which this Commitment applies placed on the market by ACEA members will have equipment allowing the use of Daytime Running Lights (DRL).

³ OJ L42, 23.2.1970, p. 1.

⁴ OJ L11, 16.1.1999, p. 25.

15. ACEA supports the objectives of the e-Europe action plan and agrees on the importance of additional ICT elements in improving active safety. Joint efforts will be continued in the framework of the IST and GROWTH programmes. An indicative list of possible measures is annexed to this Commitment.

ACEA members commit themselves to install progressively additional active safety devices in all new motor vehicles.

A first road map on work ahead will be presented by industry in 2002.

Bull-bars

16. From 1 January 2002, bull bars installed as original equipment on new motor vehicles to which this Commitment applies, or sold as spare parts, will not be of a rigid nature (as defined in Annex I).

MONITORING OF COMPLIANCE

17. ACEA member companies will report on their compliance with this Commitment to a Monitoring Committee set up by the Commission with the participation of ACEA representatives.
18. Compliance with this Commitment will be verified by independent technical services capable to conduct the tests defined in its annexes. ACEA members will use technical services which satisfy the harmonised standards on the operation of technical laboratories EN 45001 for the purpose of verification of compliance with this Commitment.
19. ACEA and its members will make every effort to ensure that a sufficient number of technical services are capable of conducting the tests provided in Annex III of this Commitment. A list of these technical services will be provided to the Monitoring Committee by no later than 31 December 2002. Changes to the list will be communicated to the Monitoring Committee.
20. As from the dates specified in paragraphs 8, 9, 10 and 11, ACEA member companies will supply the Monitoring Committee with copies of the certified test reports delivered by the technical services regarding compliance of new types of motor vehicles and new motor vehicles with this Commitment.
21. As from 2002, ACEA will, once a year, report to the Monitoring Committee on the progress made towards achieving compliance with the Commitment.
The Commission's official reports on the monitoring results should not refer to individual companies' achievements nor descriptions of vehicle concepts and technologies.

ANNEX I

Definitions.

ANNEX II

Technical requirements.

ANNEX III

Technical test prescriptions.

ANNEX IV

Advanced active safety devices.

ANNEX I

DEFINITIONS

1. SCOPE

- 1.1. The Commitment applies to the frontal surfaces of vehicles in categories M₁ and to N₁ vehicles derived from M₁ as defined in Directive 70/156/EEC, of a total permissible mass not exceeding 2,5 tonnes.
- 1.2. This Commitment has the purpose of reducing injuries to pedestrians and other vulnerable road users who are hit by the frontal surfaces of the vehicles defined in Paragraph 1.1.

2. DEFINITIONS

- 2.1.1 When performing measurements on the vehicle as described in this Annex, the vehicle should be positioned in its normal ride attitude as described in Paragraph 2.1.3. If the vehicle is fitted with a badge, mascot or other structure which would bend back or retract under a low applied load, then such a load shall be applied before and/or while these measurements are taken. Any vehicle component which could change shape or position, such as 'pop-up' headlights, other than suspension components or active devices to protect pedestrians, shall be set to a shape or position that the test institutes in consultation with the manufacturer consider to be the most appropriate, while taking these measurements.
- 2.1.2 **'Primary reference marks'** means holes, surfaces, marks and identification signs on the vehicle body. The type of reference mark used and the vertical (Z) position of each mark relative to the ground shall be specified by the vehicle manufacturer according to the running conditions specified in Paragraph 2.1.3. These marks shall be selected such as to be able to easily check the vehicle front and rear ride heights and vehicle attitude.

If the primary reference marks are found to be within ± 25 mm of the design position in the vertical (Z) axis, then the design position shall be considered to be the normal ride height. If this condition is met, either the vehicle shall be adjusted to the design position, or all further measurements shall be adjusted, and tests performed, to simulate the vehicle being at the design position.

- 2.1.3 **'Normal ride attitude'** is the vehicle attitude in running order positioned on the ground, with the tyres inflated to the recommended pressures, the front wheels in the straight-ahead position, with maximum capacity of all fluids necessary for operation of the vehicle, with all standard equipment as provided by the vehicle manufacturer, with a 75 kg mass placed on the driver's seat and with a 75 kg mass placed on the front passenger's seat, and with the suspension set for a driving speed of 40 km/h or

35 km/h in normal running conditions specified by the manufacturer (especially for vehicles with an active suspension or a device for automatic levelling).

2.2 **'Bumper'** is for this Commitment the front, lower, outer structure of a vehicle. It includes all structures that are intended to give protection to a vehicle when involved in a low speed frontal collision with another vehicle and also any attachments to this structure. The reference height and lateral limits of the bumper are identified by the corners and the bumper reference lines as defined in Paragraph 2.2.1 to 2.2.5.

2.2.1 **'The Upper Bumper Reference Line'** identifies the upper limit to significant points of pedestrian contact with the bumper. It is defined as the geometric trace of the upper most points of contact between a straight edge 700 mm long and the bumper, when the straight edge, held parallel to the vertical longitudinal plane of the car and inclined rearwards by 20°, is traversed across the front of the car, while maintaining contact with the ground and with the surface of the bumper (see Figure 1a).

Where necessary the straight edge shall be shortened to avoid any contact with structures above the bumper.

2.2.2 **'The Lower Bumper Reference Line'** identifies the lower limit to significant points of pedestrian contact with the bumper. It is defined as the geometric trace of the lower most points of contact between a straight edge 700 mm long and the bumper, when the straight edge, held parallel to the vertical longitudinal plane of the car and inclined forwards by 25°, is traversed across the front of the car, while maintaining contact with the ground and with the surface of the bumper (see Figure 1b).

2.2.3 **'Upper Bumper Height'** is the vertical distance between the ground and the upper bumper reference line, defined in Paragraph 2.2.1 with the vehicle positioned in its normal ride attitude.

2.2.4 **'Lower Bumper Height'** is the vertical distance between the ground and the lower bumper reference line, defined in Paragraph 2.2.2 with the vehicle positioned in its normal ride attitude.

2.2.5 **'Corner of Bumper'** is defined as the vehicle's point of contact with a vertical plane which makes an angle of 60° with the vertical longitudinal plane of the car and is tangential to the outer surface of the bumper (see Figure 2).

2.2.6 **'Ground Reference Level'** is the horizontal plane parallel to the ground level, representing the ground level for a vehicle placed at rest on a flat surface with the hand brake on, with the vehicle positioned in its normal ride attitude.

2.2.7 **'Third of the bumper'** is defined as the geometric trace between the 'Corners of the bumper' as defined in Paragraph 2.2.5, measured with a flexible tape following the outer contour of the bumper, divided in three equal parts.

2.3 **'Bonnet Leading Edge'** for this Commitment is the front upper outer structure including the bonnet and wings, the upper and side members of the headlight surround and any other attachments. The reference line identifying the position of the leading edge is defined by its height above the ground and by the horizontal distance separating it from the bumper (bumper lead), determined in accordance with Paragraph 2.3.1, 2.3.2 and 2.3.3.

- 2.3.1 **'Bonnet Leading Edge Reference Line'** is defined as the geometric trace of the points of contact between a straight edge 1 000 mm long and the front surface of the bonnet, when the straight edge, held parallel to the vertical longitudinal plane of the car and inclined rearwards by 50° and with the lower end 600 mm above the ground, is traversed across and in contact with the bonnet leading edge (See Figure 3). For vehicles having the bonnet top surface inclined at essentially 50°, so that the straight edge makes a continuous contact or multiple contacts rather than a point contact, determine the reference line with the straight edge inclined rearwards at an angle of 40°. For vehicles of such shape that the bottom end of the straight edge makes first contact then that contact is taken to be the bonnet leading edge reference line, at that lateral position. For vehicles of such shape that the top end of the straight edge makes first contact then the geometric trace of 1 000 mm wrap around distance as defined in Paragraph 2.4.1, will be used as bonnet leading edge reference line at that lateral position.

The top edge of the bumper shall also be regarded as the bonnet leading edge for this Commitment, if it is contacted by the straight edge during this procedure.

- 2.3.2 **'Bonnet Leading Edge Height'** for any section of a car is the vertical distance between the ground and the bonnet leading edge reference line defined in Paragraph 2.3.1, with the vehicle positioned in its normal ride attitude.
- 2.3.3 **'Bumper Lead'** for any section of a car is the horizontal distance between the upper bumper reference line, as defined in Paragraph 2.2.1 and the bonnet leading edge reference line, as defined in Paragraph 2.3.1.
- 2.3.4 **'Third of the bonnet leading edge'** is defined as the geometric trace between the 'Corner reference points' as defined in Paragraph 2.4.4, measured with a flexible tape following the outer contour of the leading edge, divided in three equal parts.
- 2.4 **'Bonnet top'** in this Commitment is the outer structure that includes the upper surface of all outer structures except the windscreen, the A-pillars and structures rearwards of them. It therefore includes, but is not limited to, the bonnet, wings, scuttle, wiper spindle and lower windscreen frame. It is bounded by the geometric trace of the 1 000 mm wrap around distance in the front, as defined in Paragraph 2.4.1, the bonnet side reference lines as defined in Paragraph 2.4.2 and the bonnet rear reference line as defined in Paragraph 2.4.3
- 2.4.1 **'1 000 mm Wrap Around Distance'** is the geometric trace described on the top of the bonnet by one end of a 1 000 mm long flexible tape, when it is held in a vertical fore and aft plane of the car and traversed across the front of the bonnet and bumper. The tape is held taut throughout the operation with one end held in contact with the ground, vertically below the front face of the bumper and the other end held in contact with the bonnet top (see Figure 4). The vehicle is positioned in the normal ride attitude.

Similar procedures shall be followed, using alternative tapes of appropriate lengths to describe 1 500 and 2 100 mm wrap around distances.

- 2.4.2 **'Bonnet Side Reference Line'** is defined as the geometric trace of the highest points of contact between a straight edge 700 mm long and the side of a bonnet, when the straight edge, held parallel to the lateral vertical plane of the car and inclined inwards

by 45° is traversed down the side of the bonnet top, while maintaining contact with the surface of the body shell (see Figure 5).

- 2.4.3 **'Bonnet Rear Reference Line'** is defined as the geometric trace of the most rearward points of contact between a sphere and the bonnet top, as defined in Paragraph 2.4, when the sphere is traversed across the bonnet top, while maintaining contact with the windscreen or A-pillars (see Figure 6). The wiper blades and arms are removed during this process. For the tests described in Annex III part A, the diameter of the sphere is 165 mm. For the tests described in Annex III part B, the diameter of the sphere is 165 mm if the lower windscreen frame or A-pillar is located at a wrap around distance, as defined in Paragraph 2.4.1, of 1 500 mm or more from the ground and the diameter of the sphere is 130 mm when this wrap around distance is less than 1 500 mm. If the bonnet rear reference line is located at a wrap around distance of more than 2 100 mm from the ground, the bonnet rear reference line is defined by the geometric trace of the 2 100 mm wrap around distances, as defined in Paragraph 2.4.1.
- 2.4.4 **'Corner reference point'** is the intersection of the bonnet leading edge reference line and of the bonnet side reference line (see Figure 7).
- 2.4.5 **'Third of the bonnet top'** is defined as the geometric trace between the 'Bonnet side reference lines' as defined in Paragraph 2.4.2, measured with a flexible tape following the outer contour of the bonnet top, divided in three equal parts.
- 2.5 **'Head Performance Criterion (HPC)'** shall be calculated from the resultant of accelerometer time histories as the maximum (depending on t_1 and t_2) of the equation:

$$HPC = \left[\frac{1}{t_2 - t_1} \int_{t_1}^{t_2} a \, dt \right]^{2.5} (t_2 - t_1)$$

where 'a' is the resultant acceleration as a multiple of 'g' and t_1 and t_2 are the two time instants (expressed in seconds) during the impact, defining the beginning and the end of the recording for which the value of HPC is a maximum. Values of HPC for which the time interval ($t_1 - t_2$) is greater than 15 ms are ignored for the purposes of calculating the maximum value.

- 2.6 **'Windscreen'** is the frontal glazing of the vehicle which meets all the relevant requirements of Annex I to EU Directive 77/649/EEC.
- 2.7 **'Rear Windscreen Reference Line'** is defined as the geometric trace of contact between a straight edge and the upper windscreen frame, when the straight edge, held parallel to the vertical longitudinal plane of the car and inclined rearwards by 75°, is traversed across and in contact with the upper windscreen frame (see Figure 8).

3. DAYTIME RUNNING LIGHTS**3.1 Daytime Running Lights (DRL)**

3.1.1 Daytime Running Lights means a device consisting in a series of lamps facing in a forward direction used to make the vehicle more easily visible when driving during daytime.

3.1.2 To operate this function, this device can consist in:

- specific daytime running lamps or
- dipped beam headlamps or front fog lamps operating with nominal or reduced light intensity.

3.1.3 The Daytime Running Lights shall be switched on automatically whenever the vehicle engine is started or the vehicle starts moving; at least the rear position lamps shall be switched on simultaneously.

3.1.4 The Daytime Running Lights shall be switched off automatically when the headlamps are switched on, except when the latter are used to give intermittent luminous warnings at short intervals or when the Daytime Running Lights function is performed by the dipped beam headlamps energised at nominal voltage and switched on together with front and rear position lamps, rear registration plate lamp and side marker lamps/end-outline marker lamps, if they exist.

3.1.5 Where specific daytime running lamps are used to perform the Daytime Running Lights function, they shall be:

- type-approved under ECE Regulation n° 87 annexed to the 1958 Agreement and
- installed on the vehicle following the requirements of the paragraph 6.19 and related sub-paragraphs of the EU Directive 97/28/EC on the “Installation of lighting and light signalling devices on vehicles”, except the sub-paragraphs 6.19.7 and 6.19.8 whose requirements are replaced by the requirements of points 3.1.3 and 3.1.4 above.

3.2 ABS

ABS is a Category 1 anti-lock system which meets all the relevant requirements of Annex X to EU Directive 71/320/EEC, as amended by EU Directive 98/12/EC.

3.3 Rigid bull-bars

A rigid bull-bar is a frontal protection system fitted to the bumper or other parts of the vehicle, which provide protection to the bodywork at the front of the vehicle from damage in the event of a collision, these systems being made by steel or any other metal or material presenting similar behavioural characteristics (for instance aluminium, kevlar, carbon fibers and others).

For the purpose of this definition "front of the vehicles" does not include the tops of the bonnet and wings. A structure, such as a grille, protecting only lights is not a

frontal protection system, unless it weights more than 0.5 kg. However, if a structure protects both bodywork and lights then the whole of the structure forms the frontal protection system.

Figure 1a

Determination of upper bumper reference line

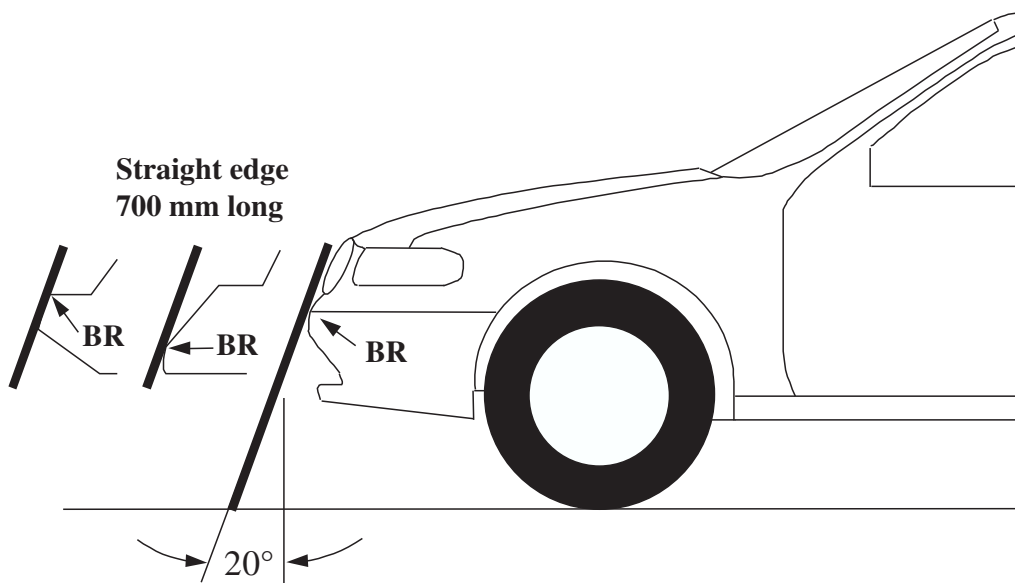


Figure 1b

Determination of lower bumper reference line

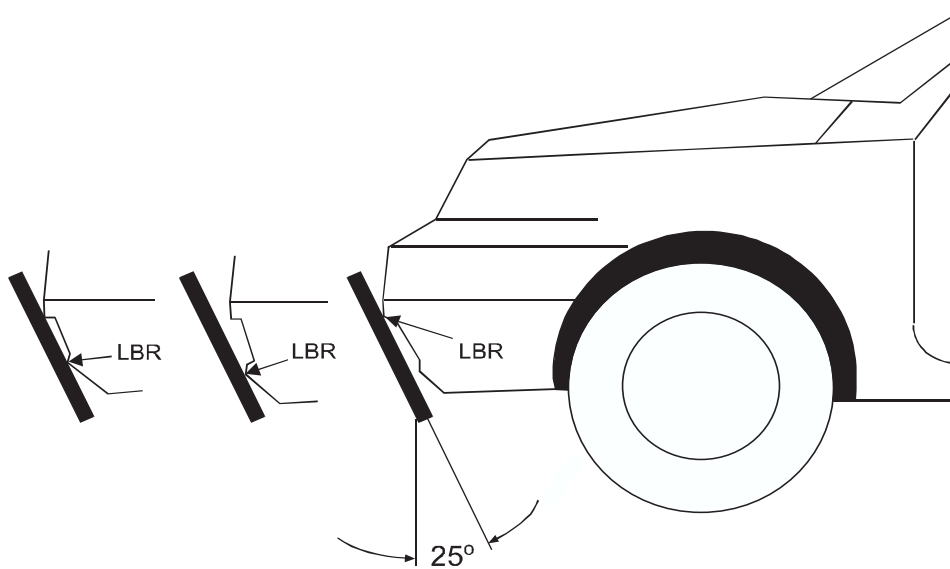
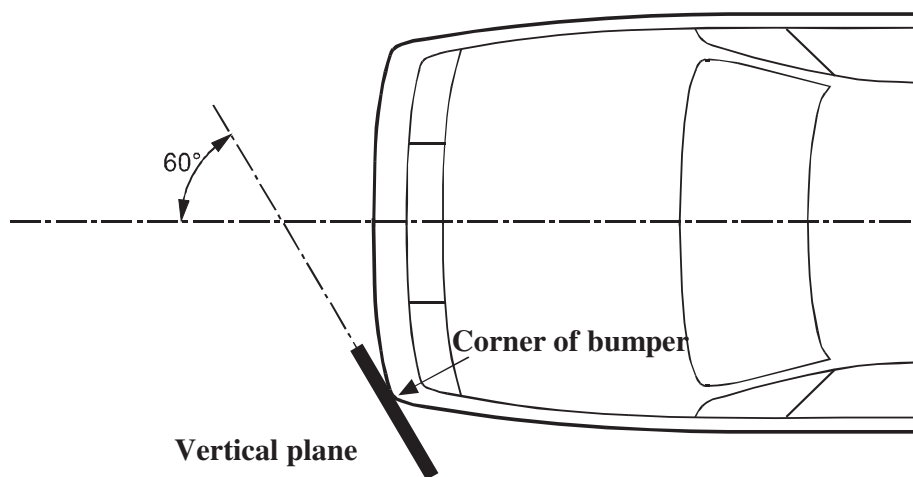


Figure 2

Determination of corner of bumper

**Figure 3**

Determination of bonnet leading edge reference line

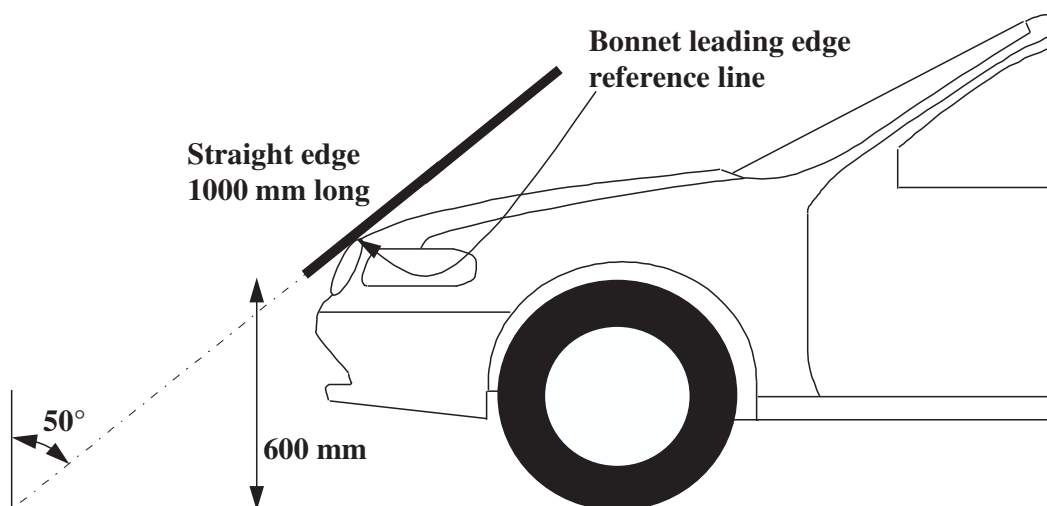
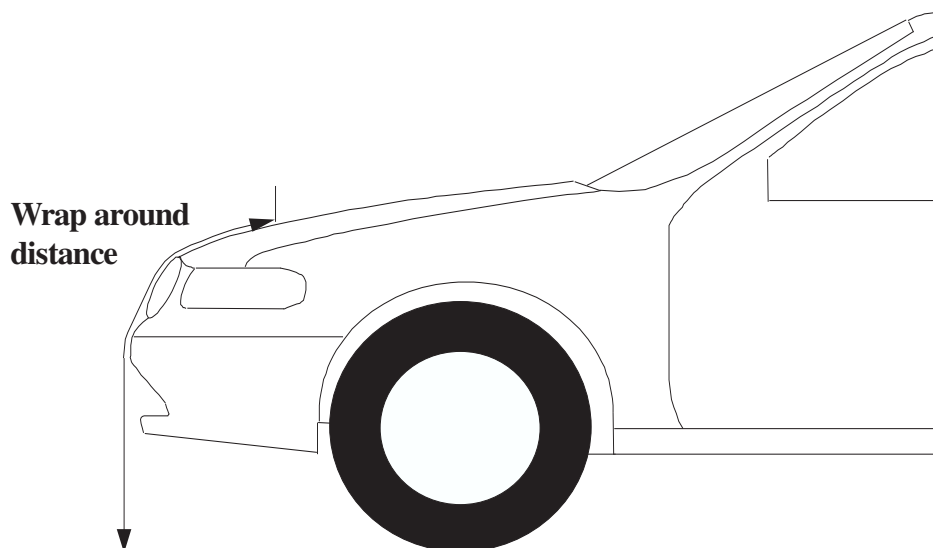


Figure 4

Determination of wrap around distance

**Figure 5**

Determination of bonnet side reference line

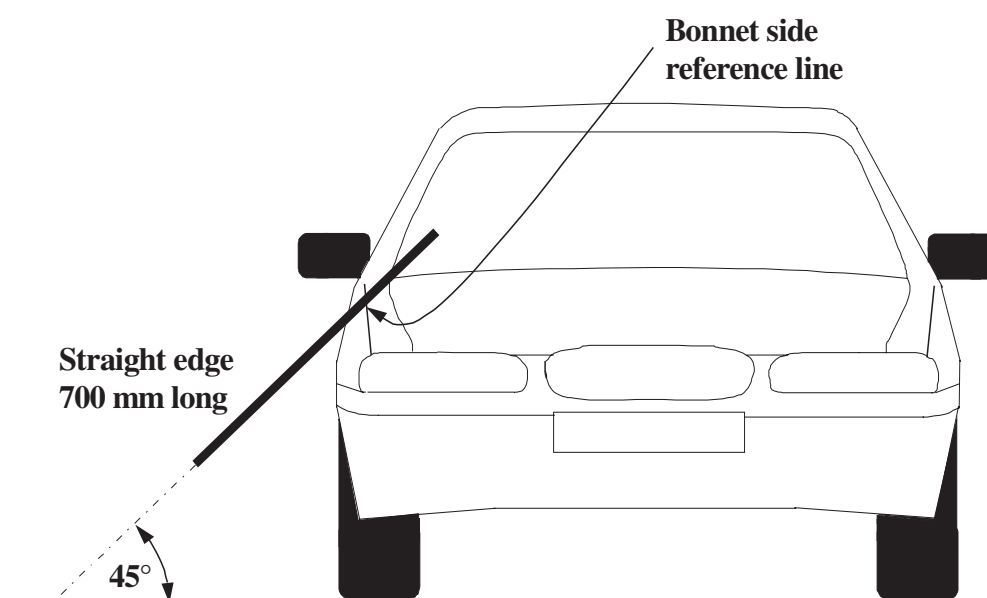


Figure 6

Determination of bonnet rear reference line

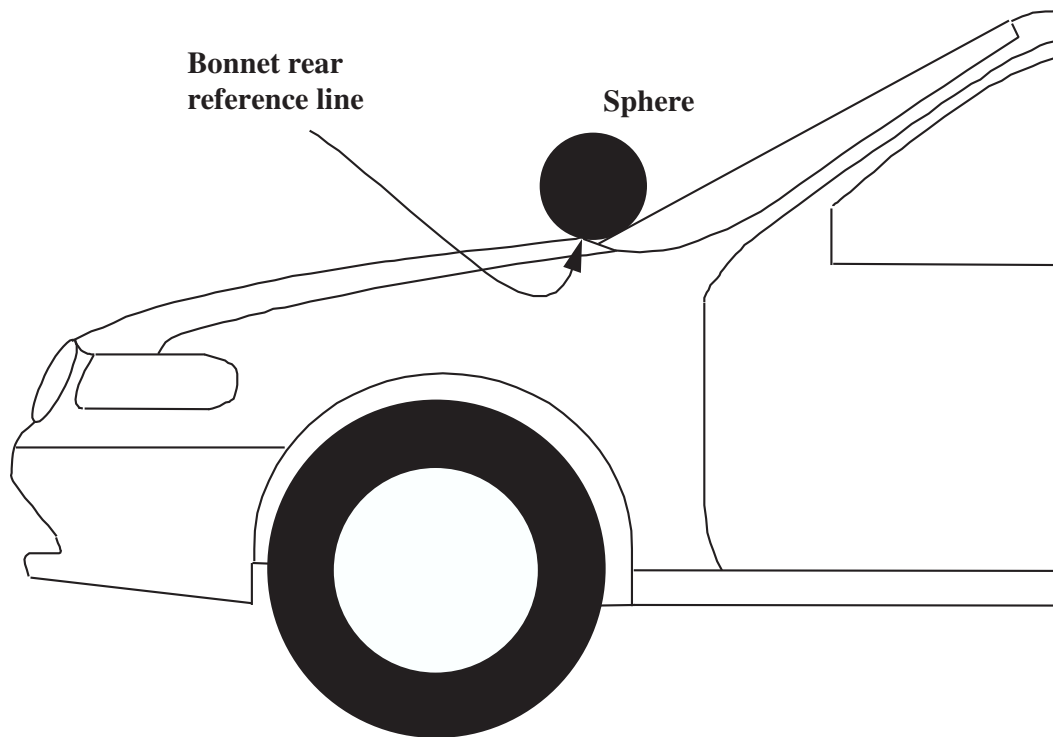


Figure 7

Determination of corner reference point; intersection of the bonnet leading edge reference line and the bonnet side reference line

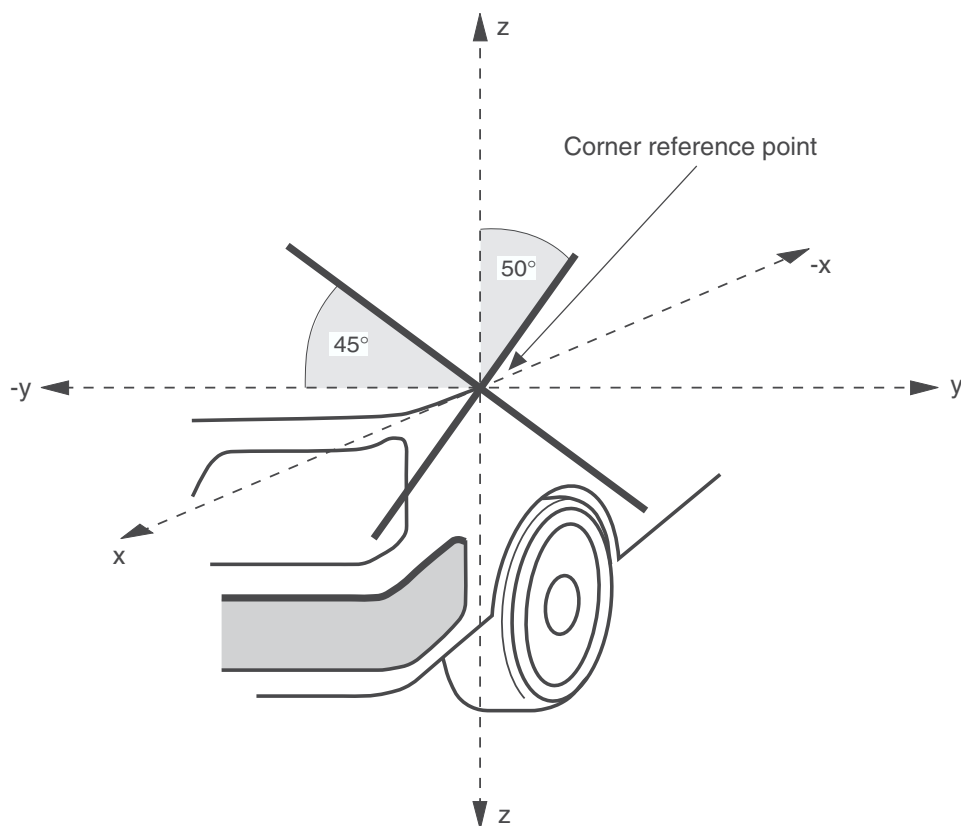
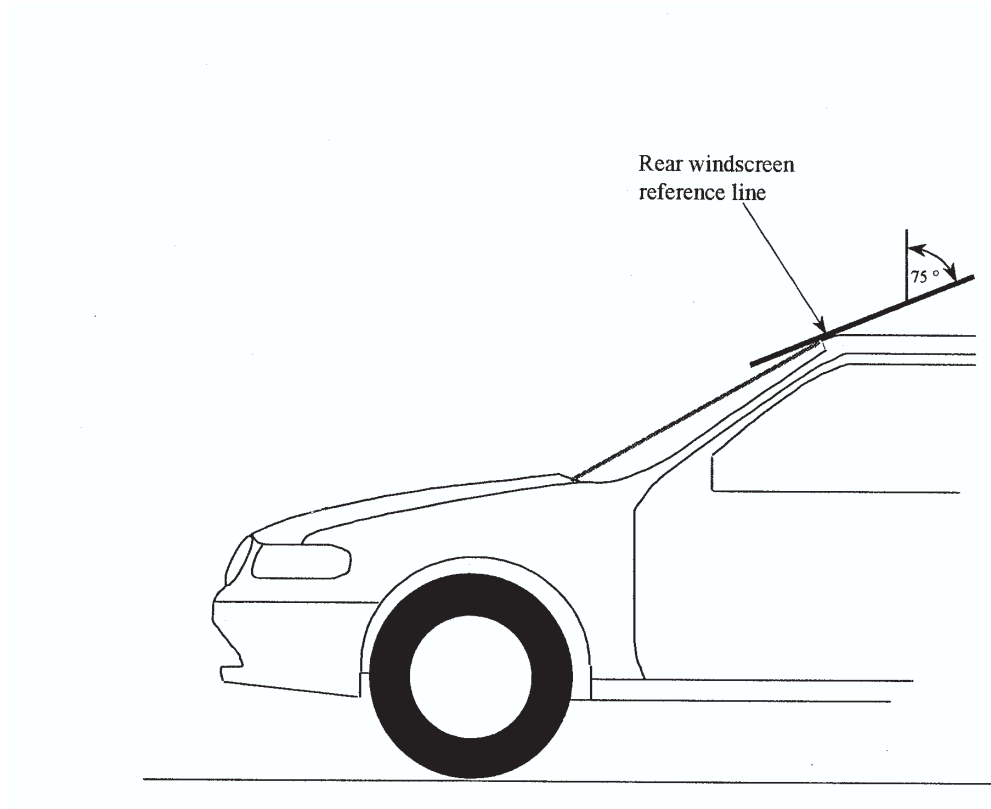


Figure 8

Determination of rear windscreen reference line



ANNEX II

TECHNICAL REQUIREMENTS

1. PART A

- 1.1 Compliance with the requirements of Paragraphs 1.2, 1.3, 1.4, 1.5 and 1.6 shall be checked in accordance with the methods set out in Annex III part A. For this purpose the test includes the initial impact and any closely following impacts which together form the main impact phase. It does not include widely spaced secondary impacts occurring after the impactor has rebounded substantially away from the vehicle.
- 1.2 In the lower legform to bumper tests at impact speeds of 40 km/h, the maximum dynamic knee bending angle shall not exceed 21.0°, the maximum dynamic knee shearing displacement shall not exceed 6.0 mm, and the acceleration measured at the upper end of the tibia shall not exceed 200 g.
- 1.3 In upper legform to bonnet leading edge tests at impact speeds up to 40 km/h, depending on the shape of the vehicle, the instantaneous sum of the impact forces with respect to time, to the top and the bottom of the impactor, shall be recorded and compared with the possible target of 5.0 kN and the bending moment on the impactor, at any of the three measuring positions, shall be recorded and compared with the possible target of 300 Nm. This test is defined as for monitoring purposes.
- 1.4 In the upper legform to bumper tests at impact speeds of 40 km/h, the instantaneous sum of the impact forces with respect to time, to the top and the bottom of the impactor, shall not exceed 7.5 kN and the bending moment on the impactor, at any of the three measuring positions, shall not exceed 510 Nm.
- 1.5 In child headform (3.5 kg impactor) to bonnet top test at impact speeds of 35 km/h, the Head Performance Criterion (HPC), calculated from the resultant of the headform accelerometer time histories, in accordance with Paragraph 2.5 of Annex I, shall not exceed 1 000 for 2/3 of the bonnet test area, 2 000 for 1/3 of the bonnet test area as defined in paragraph V 3.2 of Annex III.
- 1.6 In adult headform (4.8 kg impactor) to windscreen test at impact speeds of 35 km/h, the Head Performance Criterion (HPC), calculated from the resultant of the headform accelerometer time histories, in accordance with Paragraph 2.5 of Annex I, shall not exceed 1 000. This test is defined as for monitoring purposes.

2. PART B

- 2.1 Compliance with the requirements of Paragraphs 2.2, 2.3, 2.4 and 2.5 shall be checked in accordance with the methods set out in Annex III part B. For this purpose the tests include the initial impact and any closely following impacts which together form the main impact phase. It does not include widely spaced secondary impacts occurring after the impactor has rebounded substantially away from the vehicle.

- 2.2 In the lower legform to bumper tests at impact speeds of 40 km/h, the maximum dynamic knee bending angle shall not exceed 15.0°, the maximum dynamic knee shearing displacement shall not exceed 6.0 mm, and the acceleration measured at the upper end of the tibia shall not exceed 150 g.
- 2.3 In upper legform to bumper tests at impact speeds of 40 km/h, the instantaneous sum of the impact forces with respect to time, to the top and the bottom of the impactor, shall not exceed 5.0 kN and the bending moment on the impactor, at any of the three measuring positions, shall not exceed 300 Nm.
- 2.4 In upper legform to bonnet leading edge tests at impact speeds up to 40 km/h, depending on the shape of the vehicle, the instantaneous sum of the impact forces with respect to time, to the top and the bottom of the impactor, shall not exceed 5.0 kN and the bending moment on the impactor, at any of the three measuring positions, shall not exceed 300 Nm.
- 2.5 In adult (4.8 kg impactor) and child headform (2.5 kg impactor) to bonnet top tests at impact speeds of 40 km/h, the Head Performance Criterion (HPC), calculated from the resultant of the headform accelerometer time histories, in accordance with Paragraph 2.5 of Annex I, shall not exceed 1 000 on the whole bonnet test area

ANNEX III

CONTENTS

- I. GENERAL CONDITIONS APPLICABLE TO PART A AND B**
- II. LOWER LEGFORM TO BUMPER TESTS (PART A AND B)**
- III. UPPER LEGFORM TO BUMPER TESTS (PART A AND B)**
- IV. UPPER LEGFORM TO BONNET LEADING EDGE TESTS (PART A AND B)**
- V. CHILD HEADFORM TO BONNET TOP TESTS (PART A)**
- VI. ADULT HEADFORM TO WINDSCREEN TESTS (PART A)**
- VII. CHILD AND ADULT HEADFORM TO BONNET TOP TESTS (PART B)**

I. GENERAL CONDITIONS APPLICABLE TO PART A AND B

- 1.1 Complete Vehicle. Tests on complete vehicles shall comply with the conditions detailed in Paragraph 1.1.1, 1.1.2 and 1.1.3.
 - 1.1.1 The vehicle shall be in its normal ride attitude and shall be either securely mounted on raised supports or at rest on a flat surface with the hand brake on.
 - 1.1.2 All devices designed to protect vulnerable road users shall be correctly activated before and/or be active during the appropriate test. It shall be the responsibility of the applicant for approval to show that the devices will act as intended in a pedestrian impact.
 - 1.1.3 Any vehicle component which could change shape or position, such as ‘pop-up’ headlights, other than active devices to protect pedestrians, shall be set to a shape or position that the test institutes in consultation with the manufacturer consider to be the most appropriate, for these tests.
- 1.2 Sub-System of Vehicle. Where only a sub-system of the vehicle is supplied for tests, it shall comply with the conditions detailed in Paragraph 1.2.1, 1.2.2, 1.2.3 and 1.2.4.
 - 1.2.1 All the parts of the vehicle structure and under bonnet components or behind windscreen components that may be involved in a frontal impact with a vulnerable road user shall be included in the test to demonstrate the performance and interactions of all the contributory vehicle components.
 - 1.2.2 The vehicle sub-system shall be securely mounted in the normal vehicle ride attitude.
 - 1.2.3 All devices designed to protect vulnerable road users shall be correctly activated before and/or be active during the appropriate test. It shall be the responsibility of the applicant for approval to show that the devices will act as intended in a pedestrian impact.
 - 1.2.4 Any vehicle component which could change shape or position, such as ‘pop-up’ headlights, other than active devices to protect pedestrians, shall be set to a shape or position that the test institutes in consultation with the manufacturer consider to be the most appropriate, for these tests.

II. LOWER LEGFORM TO BUMPER TESTS

1 Scope

This test procedure is applicable to requirements both under part A and part B of the Commitment. See Annex II.

2.1 The legform impactor for the bumper tests shall be in 'free flight' at the moment of impact. The impactor shall be released to free flight at such a distance from the vehicle that the test results are not influenced by contact of the impactor with the propulsion system during rebound of the impactor.

2.2 The impactor may be propelled by an air, spring or hydraulic gun, or by other means that can be shown to give the same result.

3.1 Purpose - to test for the requirements given in Paragraphs 1.2 and 2.2 of Annex II.

3.2 A minimum of three legform to bumper tests shall be carried out, one each to the middle and the outer thirds of the bumper at positions judged to be the most likely to cause injury. Tests shall be to different types of structure, where they vary throughout the area to be assessed. The selected test points shall be a minimum of 132 mm apart, and a minimum of 66 mm inside the defined corners of the bumper. These minimum distances are to be set with a flexible tape held tautly along the outer surface of the vehicle. The positions tested by the laboratories shall be indicated in the test report.

3.3 Manufacturers might apply for derogation concerning an exemption zone for a removable towing hook. Such application should be presented to the Monitoring Committee.

3.4 Test method

3.4.1 Test apparatus

3.4.1.1 The legform impactor shall consist of two foam covered rigid segments, representing femur (upper leg) and tibia (lower leg), joined by a deformable, simulated knee joint. The overall length of the legform impactor shall be 926 ± 5 mm, having a required test mass of 13.4 ± 0.2 kg and comply with Paragraph 4 and Figure 1 of this Annex. Brackets, pulleys, etc., attached to the legform for the purpose of launching it, may extend the dimensions shown in Figure 1.

3.4.1.2 Transducers shall be fitted to measure knee bending angle and knee shearing displacement. One uniaxial accelerometer shall be fitted to the non-impacted side of the tibia, close to the knee joint, with its sensitive axis in the impact direction.

3.4.1.3 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 180 for all transducers. The CAC response values, as defined in ISO 6487: 1987, shall be 50° for the knee bending angle, 10 mm for the shearing displacement and 500 g for the acceleration. This does not require that the impactor itself be able to physically bend and shear to these angles and displacements.

3.4.1.4 The legform impactor shall meet the performance requirements specified in Paragraph 2 of Appendix I, and shall be fitted with foam cut from the sheet of material used for the dynamic certification test and fitted with deformable knee elements from the same batch as those used in the certification tests. The certified impactor may be used for a maximum of 20 impacts before re-certification. With each test new plastically deformable knee elements should be used. The impactor shall also be re-certified if more than one year has elapsed since the previous certification or if any impactor transducer output, in any impact, has exceeded the specified CAC.

3.4.1.5 The legform impactor shall be mounted, propelled and released as defined in Paragraph 2.

3.4.2 Test Procedure

3.4.2.1 The state of the vehicle or sub-system shall comply with the requirements of Paragraphs I 1.1 or 1.2 respectively. The stabilised temperature of the test apparatus and the vehicle or sub-system shall be $20^{\circ} \pm 4^{\circ}\text{C}$.

3.4.2.2 Tests shall be made to the bumper between the corners to locations defined in Paragraph 3.2.

3.4.2.3 The direction of impact shall be in the horizontal plane and parallel to the longitudinal vertical plane of the vehicle, with the axis of the legform vertical at the time of first contact. The tolerance to these directions is $\pm 2^{\circ}$.

3.4.2.4 The bottom of the legform impactor shall be at Ground Reference Level at the time of first contact with the bumper (see Figure 2), with a ± 10 mm tolerance.

When setting the height of the propulsion system, an allowance must be made for the influence of gravity during the period of free flight of the legform impactor.

At the time of first contact the impactor shall have the intended orientation about its vertical axis, for the correct operation of its knee joint, with a tolerance of $\pm 5^{\circ}$.

3.4.2.5 At the time of first contact the centre line of the legform impactor shall be within a ± 10 mm tolerance to the selected impact location.

3.4.2.6 During contact between the legform impactor and the vehicle, the legform impactor shall not contact the ground or any object not part of the vehicle.

3.4.2.7 The impact velocity of the legform impactor when striking the bumper shall be 11.1 ± 0.2 m/s. The effect of gravity shall be taken into account when the impact velocity is obtained from measurements taken before the time of first contact.

4. Legform impactor

4.1 The diameter of the femur and tibia shall be 70 ± 1 mm and both shall be covered by a 25 mm thick Confor™ foam type CF-45 and a skin made of 6 mm thick neoprene.

4.2 The 'centre of the knee' is defined as the point about which the knee effectively bends.

The 'femur' is defined as all components or parts of components (including flesh, skin covering, damper, instrumentation and brackets, pulleys, etc. attached to the legform for the purpose of launching it) above the level of the centre of the knee.

The 'tibia' is defined as all components or part of components (including flesh, skin covering, instrumentation and brackets, pulleys, etc. attached to the legform for the purpose of launching it) below the level of the centre of the knee. Note that the tibia as defined includes allowances for the mass etc. of the foot.

- 4.3 The total mass of the femur and tibia shall be 8.6 ± 0.1 kg and 4.8 ± 0.1 kg respectively, and the total mass of the legform shall be 13.4 ± 0.2 kg.

The centre of gravity of the femur and tibia shall be 217 ± 10 mm and 233 ± 10 mm from the centre of the knee respectively.

The moment of inertia of the femur and tibia, about a horizontal axis through the respective centre of gravity and perpendicular to the direction of impact, shall be 0.127 ± 0.010 kgm² and 0.120 ± 0.010 kgm² respectively.

- 4.4 A uniaxial accelerometer shall be mounted on the non-impacted side of the tibia, 66 ± 5 mm below the knee joint centre, with its sensitive axis in the direction of impact.
- 4.5 The impactor shall be instrumented to measure the bending angle and the shearing displacement between femur and tibia.
- 4.6 A damper for the shear displacement system is required, it can be mounted at any point on the rear face of the impactor or internally. The damper properties shall be such that the impactor meets both the static and dynamic shear displacement requirements and prevents excessive vibrations of the shear displacement system.

Figure 1

Legform impactor with skin and foam covering

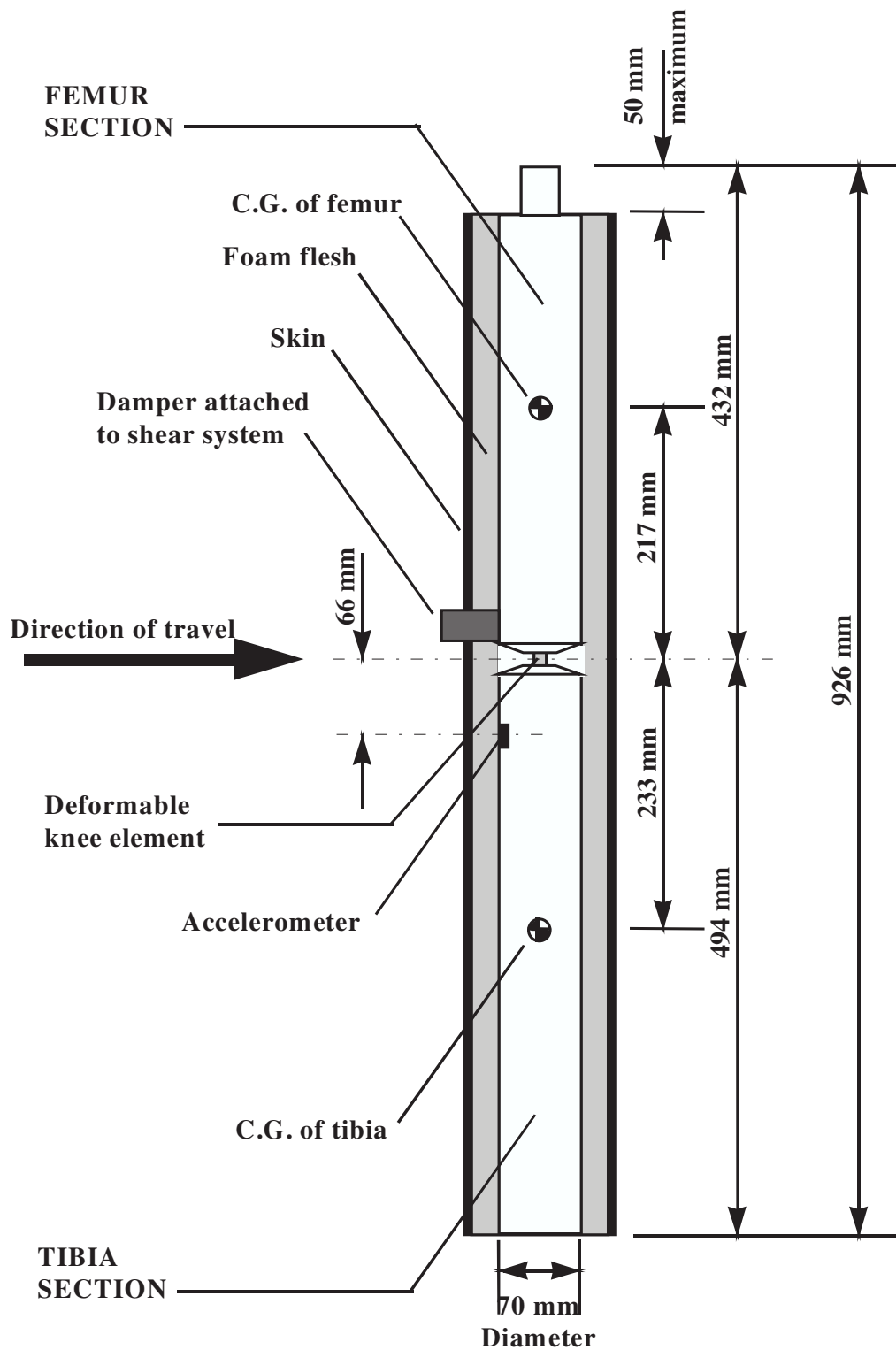
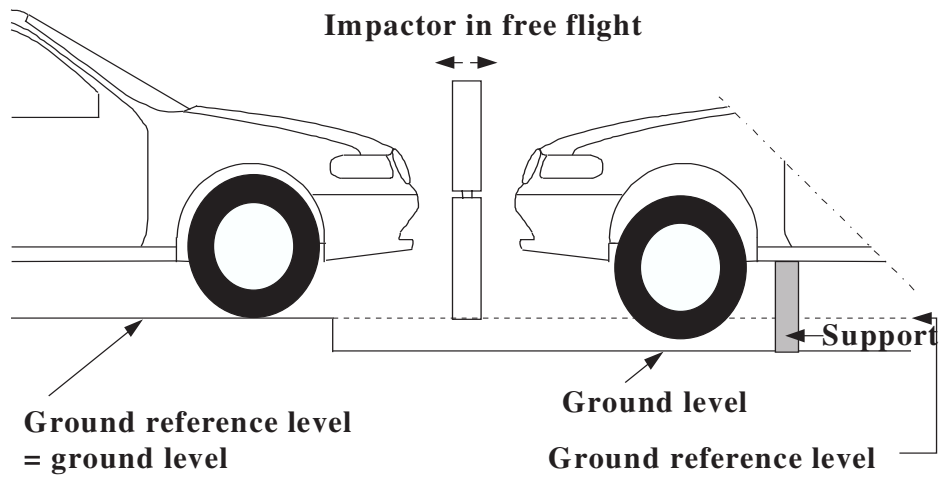


Figure 2

Legform to bumper tests for complete vehicle in normal ride attitude (left) and for complete vehicle or sub-system mounted on supports (right)



III. UPPER LEGFORM TO BUMPER TESTS

- 1.1 This test procedure is applicable to requirements both under part A and part B of the Commitment. See Annex II.
- 2.1 The upper legform impactor for the bumper test shall be mounted to the propulsion system, by a torque limiting joint, to prevent large off centre loads damaging the guidance system. The guidance system shall be fitted with low friction guides, insensitive to off-axis loading, that allow the impactor to move only in the specified direction of impact, when in contact with the vehicle. The guides shall prevent motion in other directions including rotation about any axis.
- 2.2 The impactor may be propelled by an air, spring or hydraulic gun, or by other means that can be shown to give the same result.
- 3.1 Purpose - to test for the requirements given in Paragraph 1.4 and 2.3 of Annex II.
- 3.2 Upper legform to bumper tests shall be carried out to test positions selected in Paragraph II 3.2 of this Annex, if the lower bumper height at the test position is more than 500 mm and the manufacturer elects to perform an upper legform test instead of a legform test. In exceptional cases, and only with regard to the test procedure applicable under part A, manufacturers might apply for a derogation concerning the application of this alternative test to vehicles with a lower bumper height of less than 500 mm. Such application should be presented to the Monitoring Committee.
- 3.3 Manufacturers might apply for derogation concerning an exemption zone for a removable towing hook. Such application should be presented to the Monitoring Committee.
- 3.4 Test method
 - 3.4.1 Test apparatus
 - 3.4.1.1 The upper legform impactor shall be rigid, foam covered at the impact side, and 350 ± 5 mm long and comply with Paragraph 4 and Figure 3 of this Annex.
 - 3.4.1.2 Two load transducers shall be fitted to measure individually the forces applied at either end of the upper legform impactor, plus strain gauges measuring bending moments at the centre of the upper legform impactor and at positions 50 mm either side of the centre line, see Figure 3.
 - 3.4.1.3 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 180 for all transducers. The CAC response values, as defined in ISO 6487: 1987, shall be 10 kN for the force transducers and 1 000 Nm for the bending moment measurements.
 - 3.4.1.4 The upper legform impactor shall meet the performance requirements specified in Paragraph 3 of Appendix I, and shall be fitted with foam cut from the sheet of material used for the dynamic certification test. The certified impactor may be used for a maximum of 20 impacts before re-certification (this limit does not apply to propulsion or guidance components). The impactor shall also be re-certified if more

than one year has elapsed since the previous certification or if any impactor transducer output, in any impact, has exceeded the specified CAC.

- 3.4.1.5 The upper legform impactor shall be mounted and propelled as specified in Paragraph 2.

3.4.2 Test Procedure

- 3.4.2.1 The state of the vehicle or sub-system shall comply with the requirements of Paragraphs I 1.1 or 1.2 respectively. The stabilised temperature of the test apparatus and the vehicle or sub-system shall be $20^{\circ} \pm 4^{\circ}\text{C}$.

- 3.4.2.2 Tests shall be made to the bumper between the corners to locations defined in Paragraph 3.2.

- 3.4.2.3 The direction of impact shall be parallel to the longitudinal axis of the vehicle, with the axis of the upper legform vertical at the time of first contact. The tolerance to these directions is $\pm 2^{\circ}$. At the time of first contact the impactor centre line shall be midway between the upper bumper reference line and the lower bumper reference line with a ± 10 mm tolerance and laterally with the selected impact location with a tolerance of ± 10 mm.

- 3.4.2.4 The impact velocity of the upper legform impactor when striking the bumper shall be 11.1 ± 0.2 m/s.

4. Upper legform impactor

- 4.1 The total mass of the upper legform impactor including those propulsion and guidance components which are effectively part of the impactor during the impact shall be $9.5 \text{ kg} \pm 0.1 \text{ kg}$. The upper legform impactor mass may be adjusted from this value by up to ± 1 kg, provided the required impact velocity is also changed using the formula:

$$V = \sqrt{\frac{1170}{M}}$$

where

V = impact velocity (m/s)

M = mass (kg), measured to an accuracy of better than $\pm 1\%$.

- 4.2 The total mass of the front member and other components in front of the load transducer assemblies, together with those parts of the load transducer assemblies in front of the active elements, but excluding the foam and skin, shall be 1.95 ± 0.05 kg.

- 4.3 The foam shall be two sheets of 25 mm thick Confor™ foam type CF-45. The skin shall be a 1.5 mm thick fibre reinforced rubber sheet. The foam and rubber skin together shall weigh 0.6 ± 0.1 kg (this excludes any reinforcement, mountings, etc. which are used to attach the rear edges of the rubber skin to the rear member). The foam and rubber skin shall be folded back towards the rear, with the rubber skin attached via spacers to the rear member so that the sides of the rubber skin are held

parallel. The foam shall be of such a size and shape that an adequate gap is maintained between the foam and components behind the front member, to avoid significant load paths between the foam and these components.

- 4.4 The front member shall be strain gauged to measure bending moments in three positions, as shown in Figure 1, each using a separate channel. The strain gauges are located on the impactor on the back of the front member. The two outer strain gauges are located 50 ± 1 mm from the impactor's symmetrical axis. The middle strain gauge is located on the symmetrical axis with a ± 1 mm tolerance.
- 4.5 The torque limiting joint shall be set so that the longitudinal axis of the front member is perpendicular to the axis of the guidance system, with a tolerance of $\pm 2^\circ$, with the joint friction torque set to a minimum of 650 Nm.
- 4.6 The centre of gravity of those parts of the impactor which are effectively forward of the torque limiting joint, including any weights fitted, shall lie on the longitudinal centre line of the impactor, with a tolerance of ± 10 mm.
- 4.7 The length between the load transducer centrelines shall be 310 ± 1 mm and the front member diameter shall be 50 ± 1 mm.

Figure 3

Upper legform impactor



IV. UPPER LEGFORM TO BONNET LEADING EDGE TESTS

- 1.1 This test procedure is applicable to requirements both under part A and part B of the Commitment. See Annex II.
- 2.1 The upper legform impactor for the bonnet leading edge test shall be mounted to the propulsion system, by a torque limiting joint, to prevent large off centre loads damaging the guidance system. The guidance system shall be fitted with low friction guides, insensitive to off-axis loading, that allow the impactor to move only in the specified direction of impact, when in contact with the vehicle. The guides shall prevent motion in other directions including rotation about any axis.
- 2.2 The impactor may be propelled by an air, spring or hydraulic gun, or by other means that can be shown to give the same result.
- 3.1 Purpose - to test for the requirements given in Paragraph 1.3 and 2.4 of Annex II.
- 3.2 A minimum of three upper legform to bonnet leading edge tests shall be carried out, one each to the middle and the outer thirds of the bonnet leading edge at positions judged to be the most likely to cause injury. However, the test point in each third shall be selected such that the required kinetic energy of impact, determined in Paragraph 3.4.2.7, exceeds 200 J, if such a point is available. Tests shall be to different types of structure, where they vary throughout the area to be assessed. The selected test points shall be a minimum of 150 mm apart, and a minimum of 75 mm inside the defined corner reference points. These minimum distances are to be set with a flexible tape held tautly along the outer surface of the vehicle. The positions tested by the laboratories shall be indicated in the test report.
- 3.4 Test method
 - 3.4.1 Test apparatus
 - 3.4.1.1 The upper legform impactor shall be rigid, foam covered at the impact side, and 350 ± 5 mm long and comply with Paragraph 4 and Figure 4 of this Annex.
 - 3.4.1.2 The upper legform impactor mass shall be dependent upon the general shape of the front of the car and determined as specified in Paragraph 2.4.2.7.
 - 3.4.1.3 Two load transducers shall be fitted to measure individually the forces applied at either end of the upper legform impactor, plus strain gauges measuring bending moments at the centre of the upper legform impactor and at positions 50 mm either side of the centre line, see Figure 4.
 - 3.4.1.4 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 180 for all transducers. The CAC response values, as defined in ISO 6487: 1987, shall be 10 kN for the force transducers and 1 000 Nm for the bending moment measurements.

- 3.4.1.5 The upper legform impactor shall meet the performance requirements specified in Paragraph 3 of Annex VII, and shall be fitted with foam cut from the sheet of material used for the dynamic certification test. The certified impactor may be used for a maximum of 20 impacts before re-certification (this limit does not apply to propulsion or guidance components). The impactor shall also be re-certified if more than one year has elapsed since the previous certification or if any impactor transducer output, in any impact, has exceeded the specified CAC.
- 3.4.1.6 The upper legform impactor shall be mounted and propelled as specified in Paragraph 2.
- 3.4.2 Test procedure
- 3.4.2.1 The state of the vehicle or sub-system shall comply with the requirements of Paragraphs I 1.1 or 1.2 respectively. The stabilised temperature of the test apparatus and the vehicle or sub-system shall be $20^{\circ} \pm 4^{\circ}\text{C}$.
- 3.4.2.2 Tests shall be made to the bonnet leading edge, between the 'corner reference points' to locations defined in Paragraph 3.2.
- 3.4.2.3 The upper legform impactor shall be aligned such that the centre line of the propulsion system and the longitudinal axis of the impacting upper legform impactor are in the fore and aft vertical plane of the section of the vehicle to be tested. The tolerances to these directions are $\pm 2^{\circ}$. At the time of first contact the impactor centre line shall be coincident with the bonnet leading edge reference line with a ± 10 mm tolerance (see Figure 5, and laterally with the selected impact location with a tolerance of ± 10 mm.
- 3.4.2.4 The required impact velocity, the direction of impact and the upper legform impactor mass shall be determined as specified in Paragraphs 3.4.2.6 and 3.4.2.7. The tolerance to the impact velocity is $\pm 2\%$ and the tolerance to the impact direction is $\pm 2^{\circ}$. The effect of gravity shall be taken into account when the impact velocity is obtained from measurements taken before the time of first contact. The upper legform impactor mass should be measured to an accuracy of better than $\pm 1\%$, and if the measured value differs from the required value then the required velocity should be adjusted to compensate, as specified in Paragraph 3.4.2.7.
- 3.4.2.5 Determination of vehicle shape:
- 3.4.2.5.1 The position of the upper bumper reference line shall be determined as defined in Paragraph 2.2.1 of Annex I.
- 3.4.2.5.2 The bonnet leading edge reference line shall be determined as defined in Paragraph 2.3.1 of Annex I.
- 3.4.2.5.3 For the section of bonnet leading edge to be tested the bonnet leading edge height and the bumper lead shall be determined as defined in Paragraphs 2.3.2 and 2.3.3 of Annex I.

3.4.2.6 Determine the required impact velocity and the direction of impact from Figures 6 and 7 with reference to the values of bonnet leading edge height and bumper lead determined in Paragraph 3.4.2.5.

3.4.2.7 The total mass of the upper legform impactor includes those propulsion and guidance components which are effectively part of the impactor during the impact, including the extra weights.

Calculate the value of the upper legform impactor mass from:

$$M = 2E / V^2$$

where

M = Mass [kg]

E = Impact Energy [J]

V = Velocity [m/s].

The required velocity shall be the value derived in Paragraph 3.4.2.6 and the energy shall be derived from Figure 8 with reference to the values of bonnet leading edge height and bumper lead determined in Paragraph 3.4.2.5.

The upper legform impactor mass may be adjusted from the calculated value by up to $\pm 10\%$, provided the required impact velocity is also changed using the above formula to maintain the same impactor kinetic energy.

3.4.2.8 Fit the required extra weights to give the calculated value of upper legform impactor mass, determined in Paragraph 3.4.2.7, to the rear of the rear member as shown in Figure 4, or to components of the guidance system which are effectively part of the impactor during the impact.

4. Description of upper legform impactor

4.1 The total mass of the front member and other components in front of the load transducer assemblies, together with those parts of the load transducer assemblies in front of the active elements, but excluding the foam and skin, shall be 1.95 ± 0.05 kg.

4.2 The foam shall be two sheets of 25 mm thick Confor™ foam type CF-45. The skin shall be a 1.5 mm thick fibre reinforced rubber sheet. The foam and rubber skin together shall weigh 0.6 ± 0.1 kg (this excludes any reinforcement, mountings, etc. which are used to attach the rear edges of the rubber skin to the rear member). The foam and rubber skin shall be folded back towards the rear, with the rubber skin attached via spacers to the rear member so that the sides of the rubber skin are held parallel. The foam shall be of such a size and shape that an adequate gap is maintained between the foam and components behind the front member, to avoid significant load paths between the foam and these components.

- 4.3 The front member shall be strain gauged to measure bending moments in three positions, as shown in Figure 4, each using a separate channel. The strain gauges are located on the impactor on the back of the front member. The two outer strain gauges are located 50 ± 1 mm from the impactor's symmetrical axis. The middle strain gauge is located on the symmetrical axis with a ± 1 mm tolerance.
- 4.4 The torque limiting joint shall be set so that the longitudinal axis of the front member is perpendicular to the axis of the guidance system, with a tolerance of $\pm 2^\circ$, with the joint friction torque set to a minimum of 650 Nm.
- 4.5 The centre of gravity of those parts of the impactor which are effectively forward of the torque limiting joint, including any weights fitted, shall lie on the longitudinal centre line of the impactor, with a tolerance of ± 10 mm.
- 4.6 The length between the load transducer centrelines shall be 310 ± 1 mm and the front member diameter shall be 50 ± 1 mm.

Figure 4

Upper legform impactor



Figure 5

Upper legform to bonnet leading edge tests

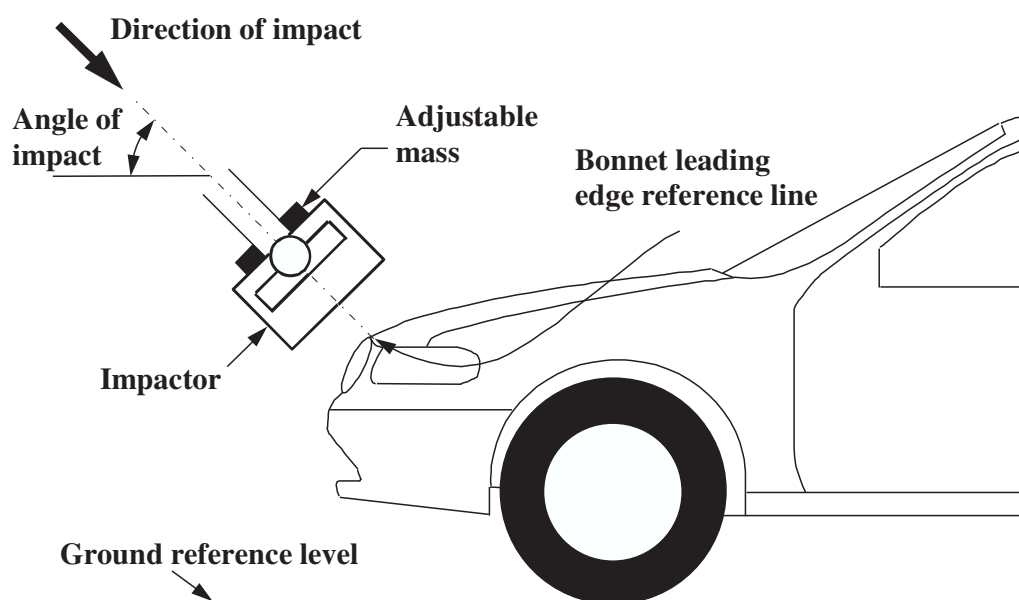
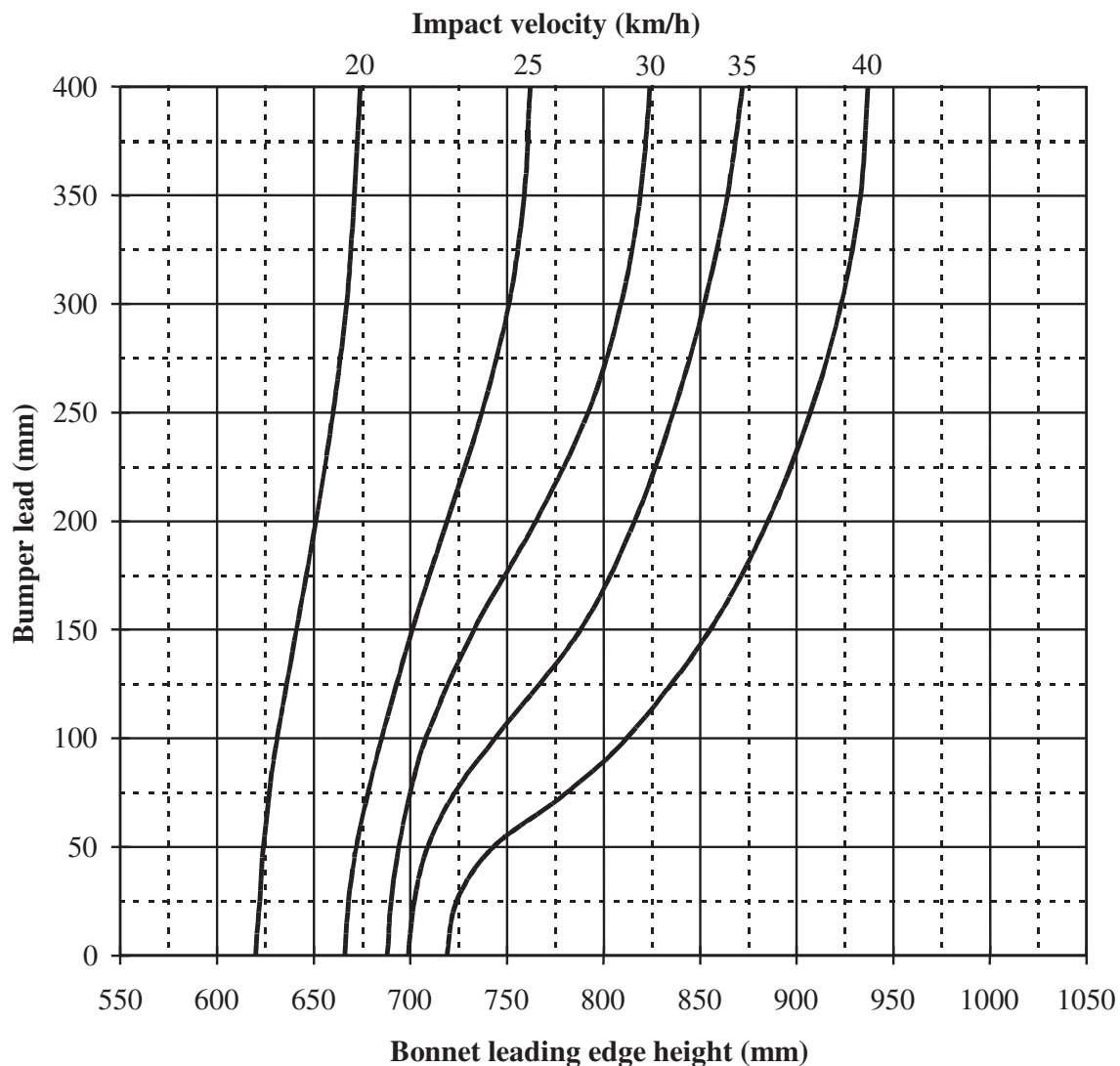


Figure 6

Velocity of upper legform to bonnet leading edge tests with respect to vehicle shape

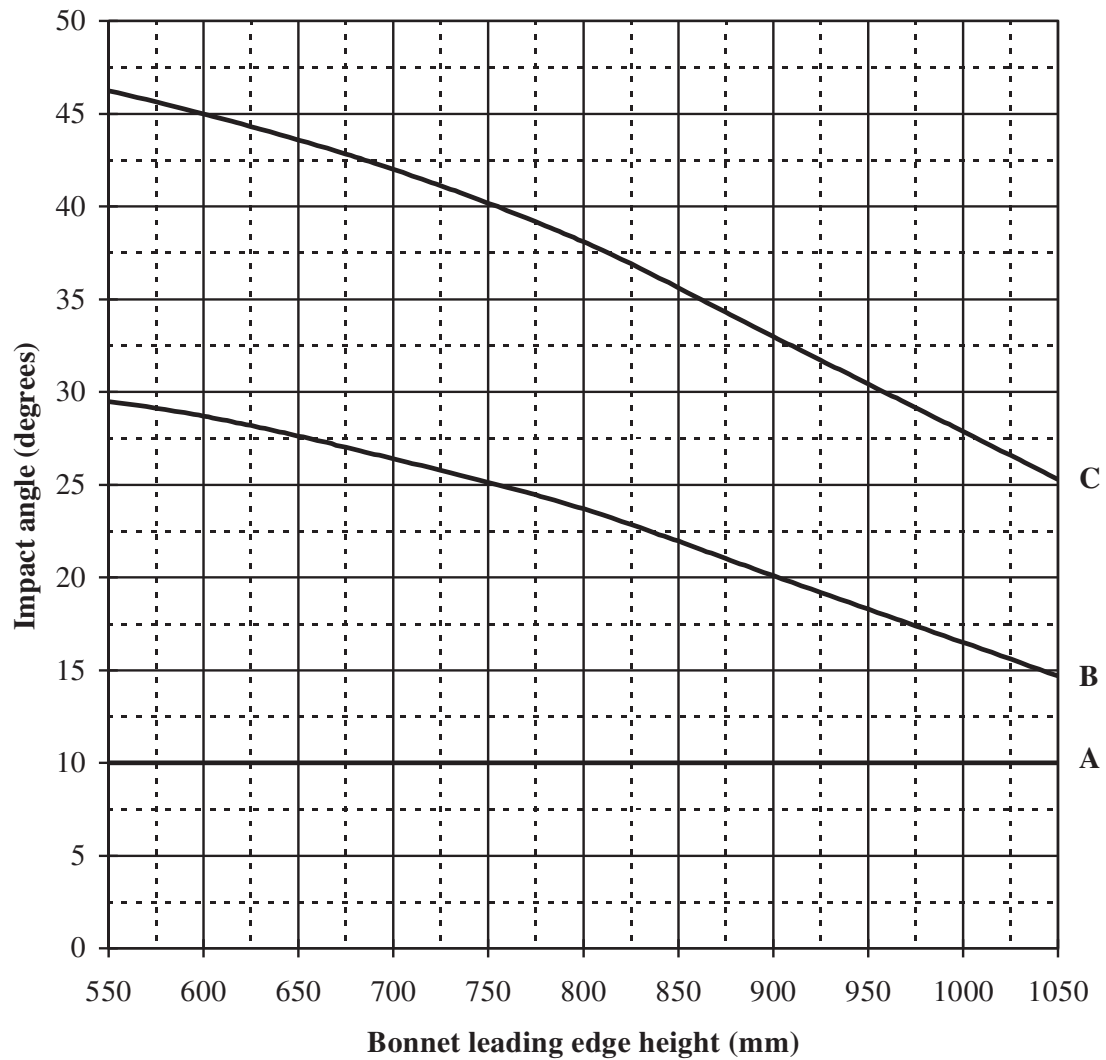


Notes:

1. Interpolate horizontally between curves.
2. With configurations below 20 km/h - test at 20 km/h.
3. With configurations above 40 km/h - test at 40 km/h.
4. With negative bumper leads - test as for zero bumper lead.
5. With bumper leads above 400 mm - test as for 400 mm.

Figure 7

Angle of upper legform to bonnet leading edge tests with respect to vehicle shape

**Key:**

A = 0 mm bumper lead

B = 50 mm bumper lead

C = 150 mm bumper lead

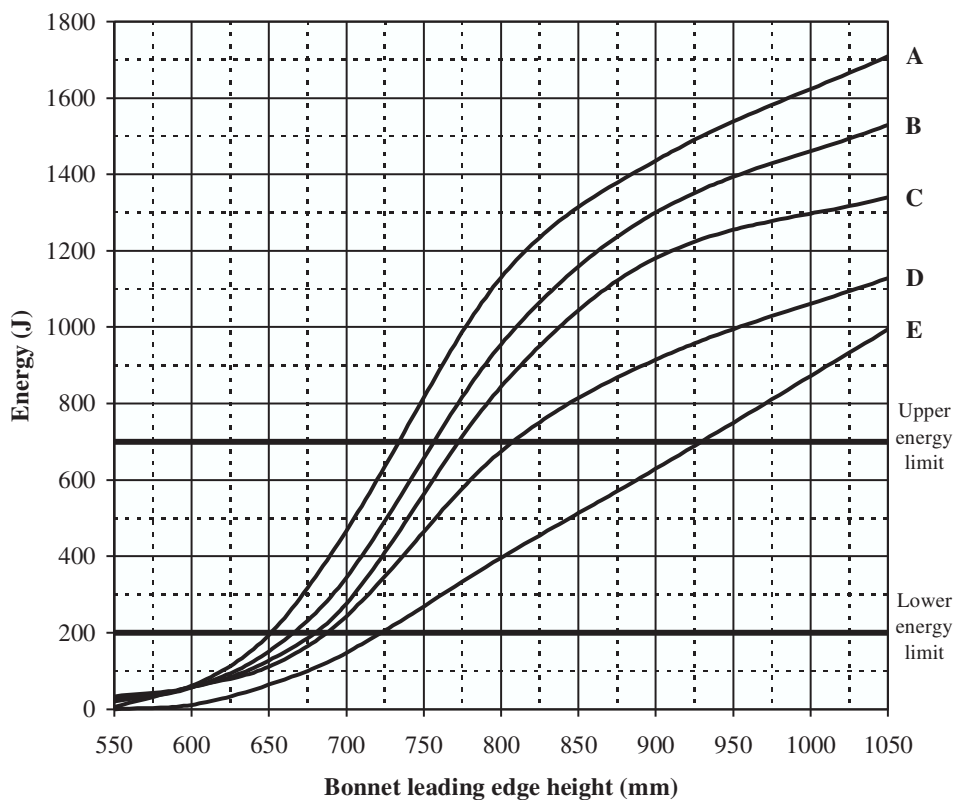
■ **■**

Notes:

- 1** Interpolate vertically between curves.
- 2** With negative bumper leads - test as for zero bumper lead.
- 3** With bumper leads above 150 mm - test as for 150 mm.
- 4** With bonnet leading edge heights above 1 050 mm - test as for 1 050 mm.

Figure 8

Kinetic energy of upper legform to bonnet leading edge tests with respect to vehicle shape

**Key:**

- A = 50 mm bumper lead
 B = 100 mm bumper lead
 C = 150 mm bumper lead
 D = 250 mm bumper lead
 E = 350 mm bumper lead

Notes:

- 1 Interpolate vertically between curves.
- 2 With bumper leads below 50 mm - test as for 50 mm.
- 3 With bumper leads above 350 mm - test as for 350 mm.
- 4 With bonnet leading edge heights above 1 050 mm - test as for 1 050 mm.
- 5 With a required kinetic energy above 700 J - test at 700 J.
- 6 With a required kinetic energy below 200 J - no test is required.

V. CHILD HEADFORM TO BONNET TOP TESTS

- 1.1 This test procedure is applicable to the requirements of part A of the Commitment.
- 2.1 The headform impactor for the bonnet top test shall be in 'free flight' at the moment of impact. The impactor shall be released to free flight at such a distance from the vehicle that the test results are not influenced by contact of the impactor with the propulsion system during rebound of the impactor.
- 2.2 The impactor may be propelled by an air, spring or hydraulic gun, or by other means that can be shown to give the same result.
- 3.1 Purpose - to test for the requirements given in Paragraph 1.5 of Annex II.
- 3.2 Headform impactor tests shall be to the bonnet top as defined in Paragraph 2.4 of Annex I. A minimum of eighteen tests shall be carried out with the headform impactor, six tests each to the middle and the outer thirds of the bonnet top, as described in Paragraph 2.4.5 of Annex I, at positions judged to be the most likely to cause injury. Tests shall be to different types of structure, where these vary throughout the area to be assessed.

Among the minimum of eighteen tests, at least twelve tests shall be carried out with the headform impactor within the “Bonnet Top Zone A” and a minimum of six tests shall be carried out within the “Bonnet Top Zone B” as defined in paragraph 3.3.

The test points shall be located so that the impactor is not expected to impact the bonnet top with a glancing blow and then impact the windscreen or an A pillar more severely. The selected test points for the child headform impactor shall be a minimum of 165 mm apart, a minimum of 82.5 mm inside the defined bonnet side reference lines, a minimum of 82.5 mm forwards of the defined bonnet rear reference line. Each selected test point for the child headform shall also be a minimum of 165 mm rearwards of the bonnet leading edge reference line, unless no point in the bonnet leading edge test area within 165 mm laterally would, if chosen for an upper legform to bonnet leading edge test, require a kinetic energy of impact of more than 200 J.

These minimum distances are to be set with a flexible tape held tautly along the outer surface of the vehicle. If a number of test positions have been selected in order of potential to cause injury and the test area remaining is too small to select another test position while maintaining the minimum spacing between tests, then less than eighteen tests may be performed. The positions tested by the laboratories shall be indicated in the test report.

However, the technical services conducting the tests shall perform as many tests as necessary to guarantee the compliance of the vehicle with the Head Protection Criteria (HPC) limit values of 1 000 for “Bonnet Top Zone A” and 2 000 for “Bonnet Top Zone B”, especially in the points near to the borders between the two types of zones.

3.3 “Bonnet Top Zone A” and “Bonnet Top Zone B”

- 3.3.1 The manufacturer shall identify the zones of the bonnet top where the Head Performance Criterion (HPC) must not exceed 1 000 (Bonnet Top Zone A) respectively 2 000 (Bonnet Top Zone B), according to the technical requirements set up in paragraph 1.5 of Annex II (see Figure 9).

Figure 9

Bonnet Top Zone A and Bonnet Top Zone B

- 3.3.2 Marking of the “Bonnet Top” impact area as well as “Bonnet Top Zone A” and “Bonnet Top Zone B” will be based on a drawing supplied by the manufacturer, when viewed from a horizontal plane above the vehicle that is parallel to the vehicle horizontal zero plane. A sufficient number of x and y co-ordinates shall be supplied by the manufacturer to mark up the areas on the actual vehicle while considering the vehicle outer contour in the z direction.
- 3.3.3 The areas of “Bonnet Top Zone A” and “Bonnet Top Zone B” may consist of several parts, with the number of these parts not being limited.
- 3.3.4 The calculation of the surface of the impact area as well as the surface areas of “Bonnet Top Zone A” and “Bonnet Top Zone B” shall be done on the basis of a projected bonnet when viewed from a horizontal plane parallel to the horizontal zero plane above the vehicle, on the basis of the drawing data supplied by the manufacturer.
- ### 3.4 Test method
- 3.4.1 Test apparatus
- 3.4.1.1 The child headform impactor shall be a rigid spheres fitted with a vinyl skin and shall comply with Paragraph 4, and with Figure 10 of this Annex. The diameter shall be 165 ± 1 mm as shown in Figure 10. The total impactor mass 3.5 ± 0.07 kg.

- 3.4.1.2 One triaxial (or three uniaxial) accelerometer shall be mounted in the centre of the sphere.
- 3.4.1.3 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 1 000. The CAC response value, as defined in ISO 6487: 1987, shall be 500 g for the acceleration.
- 3.4.1.4 The child headform impactor shall meet the performance requirements specified in Paragraph 4 of Appendix I. The certified impactor may be used for a maximum of 20 impacts before re-certification. The impactor shall be re-certified if more than one year has elapsed since the previous certification or if the transducer output, in any impact, has exceeded the specified CAC.
- 3.4.1.5 The headform impactors shall be mounted, propelled and released as specified in Paragraph 2.

3.4.2 Test procedure

- 3.4.2.1 The state of the vehicle or sub-system shall comply with the requirements of Paragraphs I 1.1 or 1.2 respectively. The stabilised temperature of the test apparatus and the vehicle or sub-system shall be $20^{\circ} \pm 4^{\circ}\text{C}$.
- 3.4.2.2 Tests shall be made to the bonnet top within the boundaries as defined in Paragraph 2.4 of Annex I, and as specified in Paragraphs 3.2, 3.4.2.3 and 3.4.2.4.

For tests at the rear of the bonnet top the headform impactor shall not contact the windscreen or A pillar before impacting the bonnet top.

- 3.4.2.3 A child headform impactor as defined in Paragraph 3.4.1 shall be used for tests to the bonnet top, with the points of first contact lying between boundaries described by wrap around distances of 1 000 mm and or by the rear of the bonnet top as defined in Paragraph 2.4.3 of Annex II.

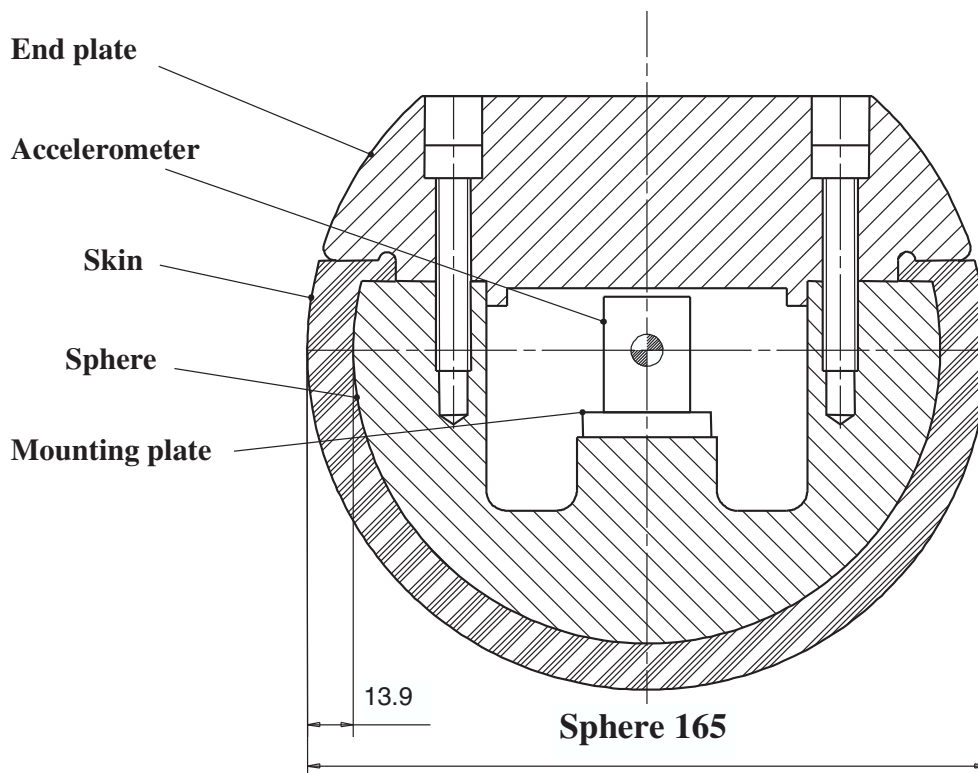
The direction of impact shall be as specified in Paragraph 3.4.2.5 and the impact velocity as specified in Paragraph 3.4.2.7.

- 3.4.2.5 The direction of impact shall be in the fore and aft vertical plane of the section of the vehicle to be tested. The tolerance for this direction is $\pm 2^{\circ}$. The direction of impact of tests to the bonnet top shall be downward and rearward, as if the vehicle were on the ground. The angle of impact for tests with the child headform impactor shall be $50^{\circ} \pm 2^{\circ}$ to the Ground Reference Level. The effect of gravity shall be taken into account when the impact angle is obtained from measurements taken before the time of first contact.
- 3.4.2.6 At the time of first contact, the point of first contact of the headform impactor shall be within a ± 10 mm tolerance to the selected impact location.
- 3.4.2.7 The impact velocity of the headform impactor when striking the bonnet top shall be 9.7 ± 0.2 m/s. The effect of gravity shall be taken into account when the impact velocity is obtained from measurements taken before the time of first contact.

4. Child Headform Impactor
- 4.1 The child headform impactor is a sphere made of aluminium and of homogenous construction.
- 4.2 The sphere shall be covered with a 13.9 ± 0.5 mm thick vinyl skin, which shall cover at least half of the sphere.
- 4.3 The centre of gravity of the child headform impactor, including instrumentation, shall be located in the centre of the sphere with a tolerance of ± 10 mm. The moment of inertia about an axis through the centre of gravity and perpendicular to the direction of impact shall be 0.0100 ± 0.0050 kgm².
- 4.4 A recess in the sphere shall allow for mounting one triaxial or three uniaxial accelerometers within a tolerance of ± 10 mm to the centre of the sphere of the child headform impactor.

Figure 10

Child headform impactor (dimensions in mm)



VI. ADULT HEADFORM TO WINDSCREEN TESTS

- 1.1 This test procedure is applicable to requirements under part A of the Commitment.

- 2.1 The headform impactor for the windscreen top test shall be in 'free flight' at the moment of impact. The impactor shall be released to free flight at such a distance from the vehicle that the test results are not influenced by contact of the impactor with the propulsion system during rebound of the impactor.
- 2.2 The impactor may be propelled by an air, spring or hydraulic gun, or by other means that can be shown to give the same result.
- 3.1 Purpose - to test for the requirements given in Paragraph 1.6 of Annex II.
- 3.2 The adult headform impactor tests shall be to the windscreen. A minimum of five tests shall be carried out with the headform impactor at positions judged to be the most likely to cause injury.

The selected test points for the adult headform impactor to the windscreen shall be a minimum of 165 mm apart, a minimum of 82.5 mm inside the windscreen limits as defined in EU Directive 77/649 EEC and a minimum of 82.5 mm forwards of the rear windscreen reference line as defined in paragraph 2.7 of Annex I (see Figure 11).

These minimum distances are to be set with a flexible tape held tautly along the outer surface of the vehicle. If a number of test positions have been selected in order of potential to cause injury and the test area remaining is too small to select another test position while maintaining the minimum spacing between tests, then less than five tests may be performed. The positions tested by the laboratories shall be indicated in the test report.

3.4 Test method

3.4.1 Test apparatus

- 3.4.1.1 The adult headform impactor shall be rigid spheres fitted with a vinyl skin and shall comply with Paragraph 4, and with Figure 12 of this Annex. The diameter shall be 165 ± 1 mm, as shown in Figure 12. The total impactor mass, including instrumentation, shall be 4.8 ± 0.1 kg.
- 3.4.1.2 One triaxial (or three uniaxial) accelerometer shall be mounted in the centre of the sphere for both child and adult headform impactors.
- 3.4.1.3 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 1 000. The CAC response value, as defined in ISO 6487: 1987, shall be 500 g for the acceleration.
- 3.4.1.4 The headform impactors shall meet the performance requirements specified in Paragraph 4 of Appendix I. The certified impactor may be used for a maximum of 20 impacts before re-certification. The impactor shall be re-certified if more than one year has elapsed since the previous certification or if the transducer output, in any impact, has exceeded the specified CAC.
- 3.4.1.5 The headform impactors shall be mounted, propelled and released as specified in Paragraph 2.

3.4.2 Test procedure

- 3.4.2.1 The state of the vehicle or sub-system shall comply with the requirements of Paragraphs I 1.1 or 1.2 respectively. The stabilised temperature of the test apparatus and the vehicle or sub-system shall be $20^{\circ} \pm 4^{\circ}\text{C}$.
- 3.4.2.2 The direction of impact shall be in the fore and aft vertical plane of the section of the vehicle to be tested. The tolerance for this direction is $\pm 2^{\circ}$. The angle of impact shall be $35^{\circ} \pm 2^{\circ}$ to the Ground Reference Level. The effect of gravity shall be taken into account when the impact angle is obtained from measurements taken before the time of first contact.
- 3.4.2.3 At the time of first contact, the point of first contact of the headform impactor shall be within a ± 10 mm tolerance to the selected impact location.
- 3.4.2.4 The impact velocity of the headform impactor when striking the windscreen shall be 9.7 ± 0.2 m/s. The effect of gravity shall be taken into account when the impact velocity is obtained from measurements taken before the time of first contact.

4. Adult Headform impactor

- 4.1 The adult headform impactor is a sphere made of aluminium and of homogenous construction.
- 4.2 The sphere shall be covered with a 12.5 ± 0.5 mm thick vinyl skin, which shall cover at least half of the sphere.
- 4.3 The centre of gravity of the adult headform impactor, including instrumentation, shall be located in the centre of the sphere with a tolerance of ± 10 mm. The moment of inertia about an axis through the centre of gravity and perpendicular to the direction of impact shall be 0.0125 ± 0.0010 kgm².
- 4.4. A recess in the sphere shall allow for mounting one triaxial or three uniaxial accelerometers within a tolerance of ± 10 mm to the centre of the sphere of the adult headform impactor.

Figure 11

Windscreen impact area

VII CHILD AND ADULT HEADFORM TO BONNET TOP TESTS

- 1.1 This test procedure is applicable to part B of the Commitment.
- 2.1 The headform impactor for the bonnet top test shall be in 'free flight' at the moment of impact. The impactor shall be released to free flight at such a distance from the vehicle that the test results are not influenced by contact of the impactor with the propulsion system during rebound of the impactor.
- 2.2 The impactor may be propelled by an air, spring or hydraulic gun, or by other means that can be shown to give the same result.
- 3.1 Purpose - to test for the requirements given in Paragraph 2.5 of Annex I.
- 3.2 Headform impactor tests shall be to the bonnet top as defined in Paragraph 2.4 of Annex I. Tests to the forward section of the bonnet top defined in Paragraph 3.4.2.3 shall be with a child headform impactor defined in Paragraph 3.4.1.1. Tests to the rearward section of the bonnet top defined in Paragraph 3.4.2.4 shall be with an adult headform impactor, defined in Paragraph 3.4.1.1. A minimum of nine tests shall be carried out with each headform impactor, three tests each to the middle and the outer thirds of the forward and rearward bonnet top sections, as described in Paragraph 2.4.5 of Annex I, at positions judged to be the most likely to cause injury. Tests shall be to different types of structure, where these vary throughout the area to be assessed.
- 3.3 The selected test points for the adult headform impactor shall be a minimum of 165 mm apart, a minimum of 82.5 mm inside the defined bonnet side reference lines and a minimum of 82.5 mm forwards of the defined bonnet rear reference line. The test points shall be located so that the impactor is not expected to impact the bonnet top with a glancing blow and then impact the windscreen or an A pillar more severely. The selected test points for the child headform impactor shall be a minimum of 130 mm apart, a minimum of 65 mm inside the defined bonnet side reference lines, a minimum of 65 mm forwards of the defined bonnet rear reference line. Each selected test point for the child headform shall also be a minimum of 130 mm rearwards of the bonnet leading edge reference line, unless no point in the bonnet leading edge test area within 130 mm laterally would, if chosen for an upper legform to bonnet leading edge test, require a kinetic energy of impact of more than 200 J.

These minimum distances are to be set with a flexible tape held tautly along the outer surface of the vehicle. If a number of test positions have been selected in order of potential to cause injury and the test area remaining is too small to select another test position while maintaining the minimum spacing between tests, then less than nine tests may be performed. The positions tested by the laboratories shall be indicated in the test report.

3.4 Test method

3.4.1 Test apparatus

3.4.1.1 The adult and child headform impactors shall be rigid spheres fitted with a vinyl skin and shall comply with Paragraph 4, and with Figures 13 and 14 respectively of this Annex. Diameters shall be 165 ± 1 mm for the adult headform and 130 ± 1 mm for the child headform respectively, as shown in Figures 13 and 14. The total impactor masses, including instrumentation, shall be 4.8 ± 0.1 kg for the adult and 2.5 ± 0.05 kg for the child headform impactor.

3.4.1.2 One triaxial (or three uniaxial) accelerometer shall be mounted in the centre of the sphere for both child and adult headform impactors.

3.4.1.3 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 1 000. The CAC response value, as defined in ISO 6487: 1987, shall be 500 g for the acceleration.

3.4.1.4 The headform impactors shall meet the performance requirements specified in Paragraph 4 of Appendix I. The certified impactor may be used for a maximum of 20 impacts before re-certification. The impactor shall be re-certified if more than one year has elapsed since the previous certification or if the transducer output, in any impact, has exceeded the specified CAC.

3.4.1.5 The headform impactors shall be mounted, propelled and released as specified in Paragraph 2.

3.4.2 Test procedure

3.4.2.1 The state of the vehicle or sub-system shall comply with the requirements of Paragraphs 1.1 or 1.2 respectively. The stabilised temperature of the test apparatus and the vehicle or sub-system shall be $20^\circ \pm 4^\circ\text{C}$.

3.4.2.2 Tests shall be made to the bonnet top within the boundaries as defined in Paragraph 2.4 of Annex II, and as specified in Paragraphs 3.2, 3.4.2.3 and 3.4.2.4.

For tests at the rear of the bonnet top the headform impactor shall not contact the windscreen or A pillar before impacting the bonnet top.

3.4.2.3 A child headform impactor as defined in Paragraph 3.4.1 shall be used for tests to the forward section of the bonnet top, with the points of first contact lying between boundaries described by wrap around distances of 1 000 mm and 1 500 mm or by the rear of the bonnet top as defined in Paragraph 2.4 of Annex I.

The direction of impact shall be as specified in Paragraph 3.4.2.5 and the impact velocity as specified in Paragraph 3.4.2.7.

3.4.2.4 An adult headform impactor as defined in Paragraph 3.4.1 shall be used for tests to the rearward section of the bonnet top, with the points of first contact lying between boundaries described by wrap around distances of 1 500 mm and 2 100 mm or by the rear of the bonnet top as defined in Paragraph 2.4 of Annex I.

The direction of impact shall be as specified in Paragraph 3.4.2.5 and the impact velocity as specified in Paragraph 3.4.2.7.

- 3.4.2.5 The direction of impact shall be in the fore and aft vertical plane of the section of the vehicle to be tested. The tolerance for this direction is $\pm 2^\circ$. The direction of impact of tests to the bonnet top shall be downward and rearward, as if the vehicle were on the ground. The angle of impact for tests with the child headform impactor shall be $50^\circ \pm 2^\circ$ to the Ground Reference Level. For tests with the adult headform impactor the angle of impact shall be $65^\circ \pm 2^\circ$ to the Ground Reference Level. The effect of gravity shall be taken into account when the impact angle is obtained from measurements taken before the time of first contact.
- 3.4.2.6 At the time of first contact, the point of first contact of the headform impactor shall be within a ± 10 mm tolerance to the selected impact location.
- 3.4.2.7 The impact velocity of the headform impactors when striking the bonnet top shall be 11.1 ± 0.2 m/s. The effect of gravity shall be taken into account when the impact velocity is obtained from measurements taken before the time of first contact.

4. **HEADFORM IMPACTORS**

4.1. Adult headform impactor

- 4.1.1 The adult headform impactor is a sphere made of aluminium and of homogenous construction.
- 4.1.2 The sphere shall be covered with a 12.5 ± 0.5 mm thick vinyl skin, which shall cover at least half of the sphere.
- 4.1.3 The centre of gravity of the adult headform impactor, including instrumentation, shall be located in the centre of the sphere with a tolerance of ± 10 mm. The moment of inertia about an axis through the centre of gravity and perpendicular to the direction of impact shall be 0.0125 ± 0.0010 kgm².
- 4.1.4. A recess in the sphere shall allow for mounting one triaxial or three uniaxial accelerometers within a tolerance of ± 10 mm to the centre of the sphere of the adult headform impactor.

4.2. Child headform impactor

- 4.2.1 The child headform impactor is a sphere made of aluminium and of homogenous construction.
- 4.2.2 The sphere shall be covered with a 12.5 ± 0.5 mm thick vinyl skin, which shall cover at least half of the sphere.
- 4.2.3 The centre of gravity of the child headform impactor, including instrumentation, shall be located in the centre of the sphere with a tolerance of ± 10 mm. The moment of inertia about an axis through the centre of gravity and perpendicular to the direction of impact shall be 0.0036 ± 0.0003 kgm².

- 4.2.4 A recess in the sphere shall allow for mounting one triaxial or three uniaxial accelerometers within a tolerance of ± 10 mm to the centre of the sphere of the child headform impactor.

Figure 13

Adult headform impactor (dimensions in mm)

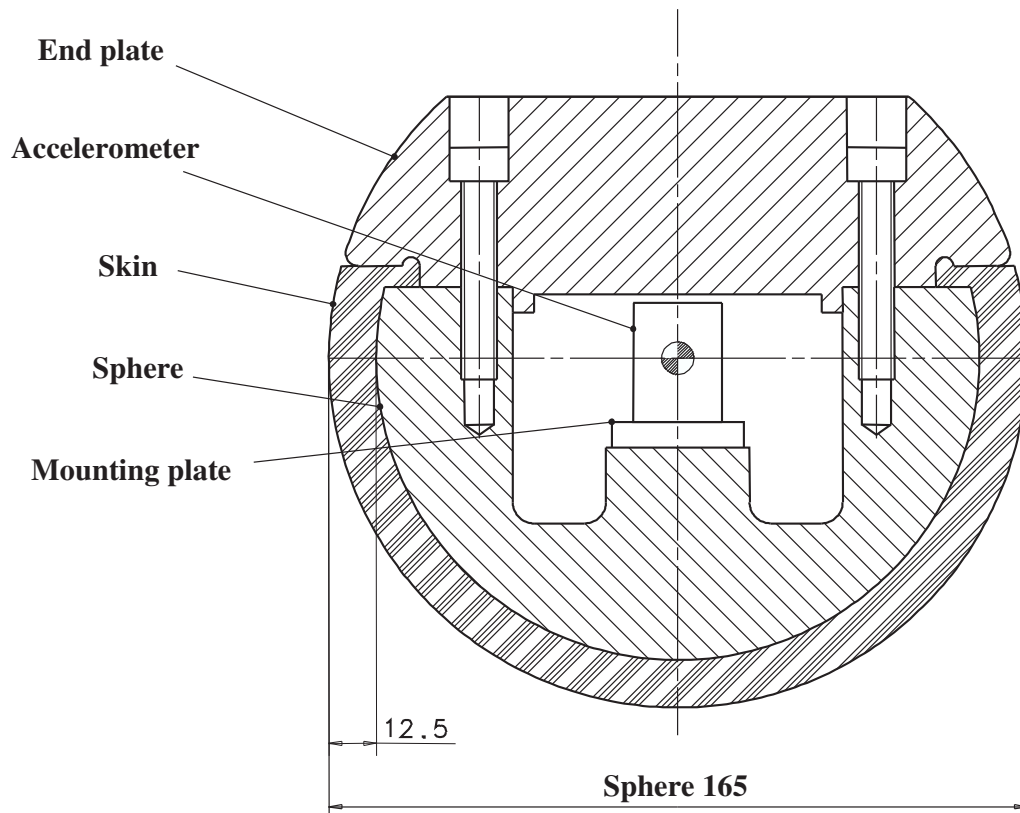
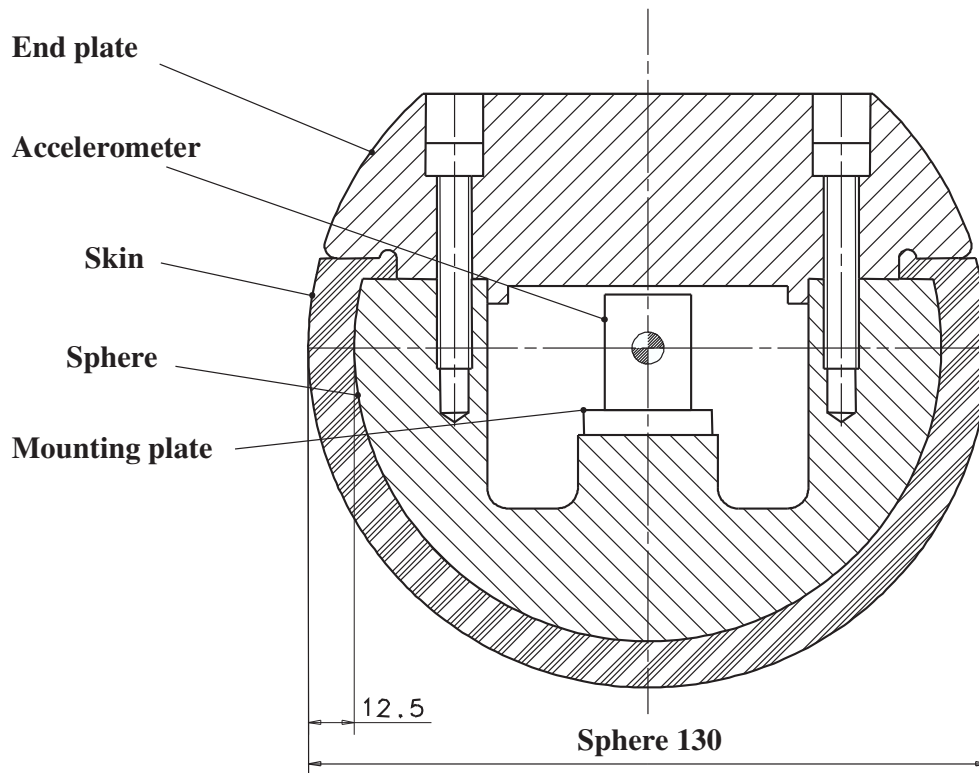


Figure 14

Child headform impactor (dimensions in mm)



APPENDIX I

CERTIFICATION OF IMPACTORS

1. CERTIFICATION REQUIREMENTS

- 1.1 The impactors that are used in the tests detailed in Annex III are required to comply with appropriate performance requirements.

The requirements for the legform impactor are specified in Paragraph 2; the upper legform impactor requirements are specified in Paragraph 3 and the adult and child headform impactor requirements are specified in Paragraph 4.

2. LEGFORM IMPACTOR

- 2.1 Static tests

- 2.1.1 The legform impactor shall meet the requirements specified in Paragraph 2.1.2 when tested as specified in Paragraph 2.1.4 and the legform impactor shall meet the requirements specified in Paragraph 2.1.3 when tested as specified in Paragraph 2.1.5.

For both tests the impactor shall have the intended orientation about its longitudinal axis, for the correct operation of its knee joint, with a tolerance of $\pm 2^\circ$.

The stabilised temperature of the impactor during certification shall be $20^\circ \pm 2^\circ\text{C}$.

The CAC response values, as defined in ISO 6487: 1987, shall be 50° for the knee bending angle and 500 N for the applied force when the legform impactor is loaded in bending in accordance with Paragraph 2.1.4, and 10 mm for the shearing displacement and 10 kN for the applied force when the legform is loaded in shearing in accordance with Paragraph 2.1.5. For both tests low-pass filtering at an appropriate frequency is permitted, to remove higher frequency noise without significantly affecting the measurement of the response of the impactor.

- 2.1.2 When the legform impactor is loaded in bending in accordance with Paragraph 2.1.4, the applied force/bending angle response shall be within the limits shown in Figure 1. Also, the energy taken to generate 15.0° of bending shall be 100 ± 7 J.
- 2.1.3 When the legform impactor is loaded in shearing in accordance with Paragraph 2.1.5, the applied force/shearing displacement response shall be within the limits shown in Figure 2.
- 2.1.4 The legform impactor, without foam covering and skin, shall be mounted with the tibia firmly clamped to a fixed horizontal surface and a metal tube connected firmly to the femur, as shown in Figure 3.

A horizontal normal force shall be applied to the metal tube at a distance of 2.0 ± 0.01 m from the centre of the knee joint and the resulting angle of knee deflection

shall be recorded. The load shall be increased until the angle of deflection of the knee is in excess of 16° .

The energy is calculated by integrating the force with respect to the bending angle in radians, and multiplying by the lever length of 2.0 ± 0.01 m.

- 2.1.5 The legform impactor, without foam covering and skin, shall be mounted with the tibia firmly clamped to a fixed horizontal surface and a metal tube connected firmly to the femur and restrained at 2.0 m from the centre of the knee joint, as shown in Figure 4.

A horizontal normal force shall be applied to the femur at a distance of 50 mm from the centre of the knee joint and the resulting knee shearing displacement shall be recorded. The load shall be increased until the shearing displacement of the knee is in excess of 8.0 mm or the load is in excess of 6.0 kN.

2.2 Dynamic tests

- 2.2.1 The legform impactor shall meet the requirements specified in Paragraph 2.2.2 when tested as specified in Paragraph 2.2.4.

The stabilised temperature of the impactor during certification shall be $20^\circ \pm 2^\circ\text{C}$.

- 2.2.2 When the legform impactor is impacted by a linearly guided certification impactor, as specified in Paragraph 2.2.4, the maximum upper tibia acceleration shall be not less than 195 g and not more than 235 g. The maximum bending angle shall be not less than 9.7° and not more than 11.7° . The maximum shearing displacement shall be not less than 5.5 mm and not more than 6.5 mm.

For all these values the readings used shall be from the initial impact with the certification impactor and not from the arresting phase. Any system used to arrest the impactor or certification impactor shall be so arranged that the arresting phase does not overlap in time with the initial impact. The arresting system shall not cause the transducer outputs to exceed the specified CAC.

- 2.2.3 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 180 for all transducers. The CAC response values, as defined in ISO 6487: 1987, shall be 50° for the knee bending angle, 10 mm for the shearing displacement and 500 g for the acceleration. This does not require that the impactor itself be able to physically bend and shear to these angles and displacements.

2.2.4 Test procedure

- 2.2.4.1 The legform impactor, including foam covering and skin, shall be suspended horizontally by three wire ropes of 1.6 ± 0.1 mm diameter and of 2.0 m minimum length, as shown in Figure 5a. It shall be suspended with its longitudinal axis horizontal, with a tolerance of $\pm 0.5^\circ$, and perpendicular to the direction of the certification impactor motion, with a tolerance of $\pm 2^\circ$. The impactor shall have the intended orientation about its longitudinal axis, for the correct operation of its knee joint, with a tolerance of $\pm 2^\circ$. The legform impactor must meet the requirements of Paragraph 3.4.1.1 of Annex III with the attachment bracket(s) for the wire ropes fitted.

- 2.2.4.2 The certification impactor shall have a mass of 16.0 ± 0.05 kg, this mass includes those propulsion and guidance components which are effectively part of the impactor during impact. The dimensions of the face of the certification impactor shall be as specified in Figure 5b. The face of the certification impactor shall be made of aluminium, with an outer surface finish of better than 2.0 micrometers.

The guidance system shall be fitted with low friction guides, insensitive to off-axis loading, that allow the impactor to move only in the specified direction of impact, when in contact with the vehicle. The guides shall prevent motion in other directions including rotation about any axis.

- 2.2.4.3 The legform impactor shall be certified with previously unused foam.
- 2.2.4.4 The legform impactor foam shall not be excessively handled or deformed before, during or after fitting.
- 2.2.4.5 The certification impactor shall be propelled horizontally at a velocity of $[7.5] \pm 0.1$ m/s into the stationary legform impactor as shown in Figure 5a. The certification impactor shall be positioned so that its centreline aligns with a position on the tibia centreline of 50 mm from the centre of the knee, with tolerances of ± 3 mm laterally and ± 3 mm vertically.

3. UPPER LEGFORM IMPACTOR

- 3.1 The upper legform impactor shall meet the requirements specified in Paragraph 3.2 when tested as specified in Paragraph 3.3.

The stabilised temperature of the impactor during certification shall be $20^\circ \pm 2^\circ\text{C}$.

3.2 Requirements

- 3.2.1 When the upper legform impactor is propelled into a stationary cylindrical pendulum the peak force measured in each load transducer shall be not less 1.20 kN and not more than 1.55 kN and the difference between the peak forces measured in the top and bottom load transducers shall not be more than 0.10 kN. Also, the peak bending moment measured by the strain gauges shall not be less than 190 Nm and not more than 250 Nm on the centre position and not less than 160 Nm and not more than 220 Nm for the outer positions. The difference between the upper and lower peak bending moments shall not be more than 20 Nm.

For all these values the readings used shall be from the initial impact with the pendulum and not from the arresting phase. Any system used to arrest the impactor or pendulum shall be so arranged that the arresting phase does not overlap in time with the initial impact. The arresting system shall not cause the transducer outputs to exceed the specified CAC.

- 3.2.2 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 180 for all transducers. The CAC response values, as defined in ISO 6487: 1987, shall be 10 kN for the force transducers and 1 000 Nm for the bending moment measurements.

3.3 Test procedure

- 3.3.1 The upper legform impactor shall be mounted to the propulsion and guidance system, by a torque limiting joint. The torque limiting joint shall be set so that the longitudinal axis of the front member is perpendicular to the axis of the guidance system, with a tolerance of $\pm 2^\circ$, with the joint friction torque set to a minimum of 650 Nm. The guidance system shall be fitted with low friction guides that allow the impactor to move only in the specified direction of impact, when in contact with the pendulum.
- 3.3.2 The impactor mass shall be adjusted to give a mass of 12 ± 0.1 kg, this mass includes those propulsion and guidance components which are effectively part of the impactor during impact.
- 3.3.3 The centre of gravity of those parts of the impactor which are effectively forward of the torque limiting joint, including the extra weights fitted, shall lie on the longitudinal centreline of the impactor, with a tolerance of ± 10 mm.
- 3.3.4 The impactor shall be certified with previously unused foam.
- 3.3.5 The impactor foam shall not be excessively handled or deformed before, during or after fitting.
- 3.3.6 The impactor with the front member vertical shall be propelled horizontally at a velocity of 7.1 ± 0.1 m/s into the stationary pendulum as shown in Figure 6.
- 3.3.7 The pendulum tube shall have a mass of 3 ± 0.03 kg, an outside diameter of 150^{+1}_{-4} mm and a wall thickness of 3 ± 0.15 mm. Total pendulum tube length shall be 275 ± 25 mm. The pendulum tube shall be made from cold drawn seamless steel (metal surface plating is permissible for protection from corrosion), with an outer surface finish of better than 2.0 micrometers. It shall be suspended on two wire ropes of 1.6 ± 0.1 mm diameter and of 2.0 m minimum length. The surface of the pendulum shall be clean and dry. The pendulum tube shall be positioned so that the longitudinal axis of the cylinder is perpendicular to the front member (i.e. level), with a tolerance of $\pm 2^\circ$, and to the direction of impactor motion, with a tolerance of $\pm 2^\circ$, and with the centre of the pendulum tube aligned with the centre of the impactor front member, with tolerances of ± 5 mm laterally and ± 5 mm vertically.

4. HEADFORM IMPACTOR

- 4.1 The child and the adult headform impactors shall meet the requirements specified in Paragraph 4.2 when tested as specified in Paragraph 4.3.

The stabilised temperature of the impactors during certification shall be $20^\circ \pm 2^\circ\text{C}$.

4.2 Requirements

- 4.2.1 When the child headform impactor is impacted by a linearly guided certification impactor, as specified in Paragraph 4.3, the peak resultant acceleration measured by one triaxial (or three uniaxial) accelerometer in the headform shall be not less than

300 g and not more than 330 g. The resultant acceleration time curve shall be uni-modal.

- 4.2.2 When the adult headform impactor is impacted by a linearly guided certification impactor, as specified in Paragraph 4.3, the peak resultant acceleration measured by one triaxial (or three uniaxial) accelerometer in the headform shall be not less than 300 g and not more than 330 g. The resultant acceleration time curve shall be uni-modal.
- 4.2.3 The instrumentation response value CFC, as defined in ISO 6487: 1987, shall be 1 000. The CAC response value, as defined in ISO 6487: 1987, shall be 500 g for the acceleration.
- 4.3 Test procedure
- 4.3.1 The headform impactor shall be suspended as shown in Figure 7. The headform impactor shall be suspended with the rear face at an angle between 25° and 90° with the horizontal, as shown in Figure 7.
- 4.3.2 The certification impactor shall have a mass of 1.0 ± 0.01 kg. This mass includes those propulsion and guidance components which are effectively part of the impactor during impact. The linear guidance system shall be fitted with low friction guides which do not contain any rotating parts. The diameter of the flat impactor face shall be 70 ± 1 mm, while the edge shall be rounded by a 5 ± 0.5 mm radius. The face of the certification impactor shall be made of aluminium, with an outer surface finish of better than 2.0 micrometers.
- 4.3.3 The certification impactor shall be propelled horizontally at a velocity of 7.0 ± 0.1 m/s into the stationary headform impactor as shown in Figure 7. The certification impactor shall be positioned so that the centre of gravity of the headform is located on the centre line of the certification impactor, with tolerances of ± 5 mm laterally and ± 5 mm vertically.
- 4.3.4 The test shall be performed on three different impact locations on the headform impactor. Previously used and/or damaged skins shall be tested in those specific areas.

Figure 1

Force versus angle requirement in static legform impactor bending certification test

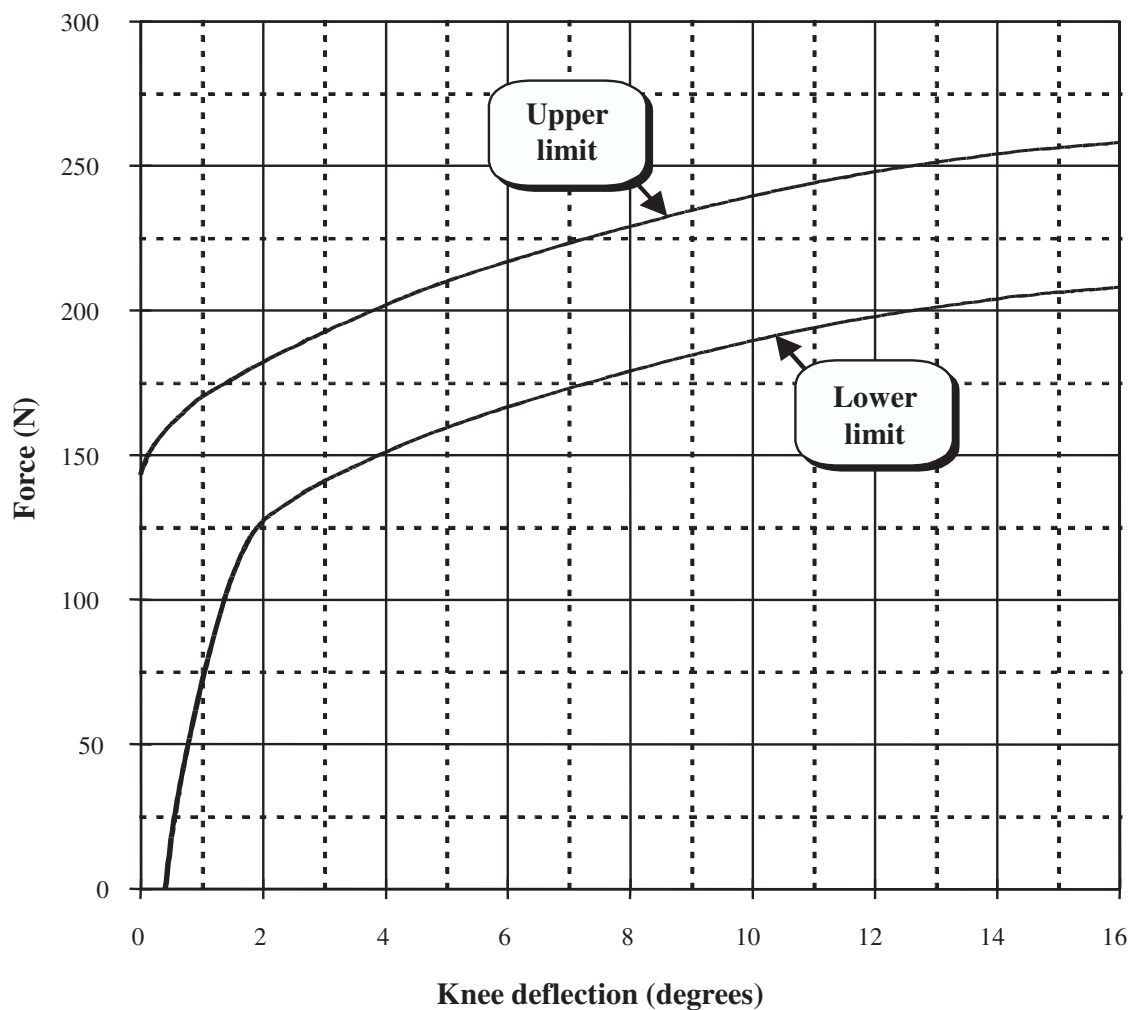


Figure 2

Force versus displacement requirement in static legform impactor shearing certification test

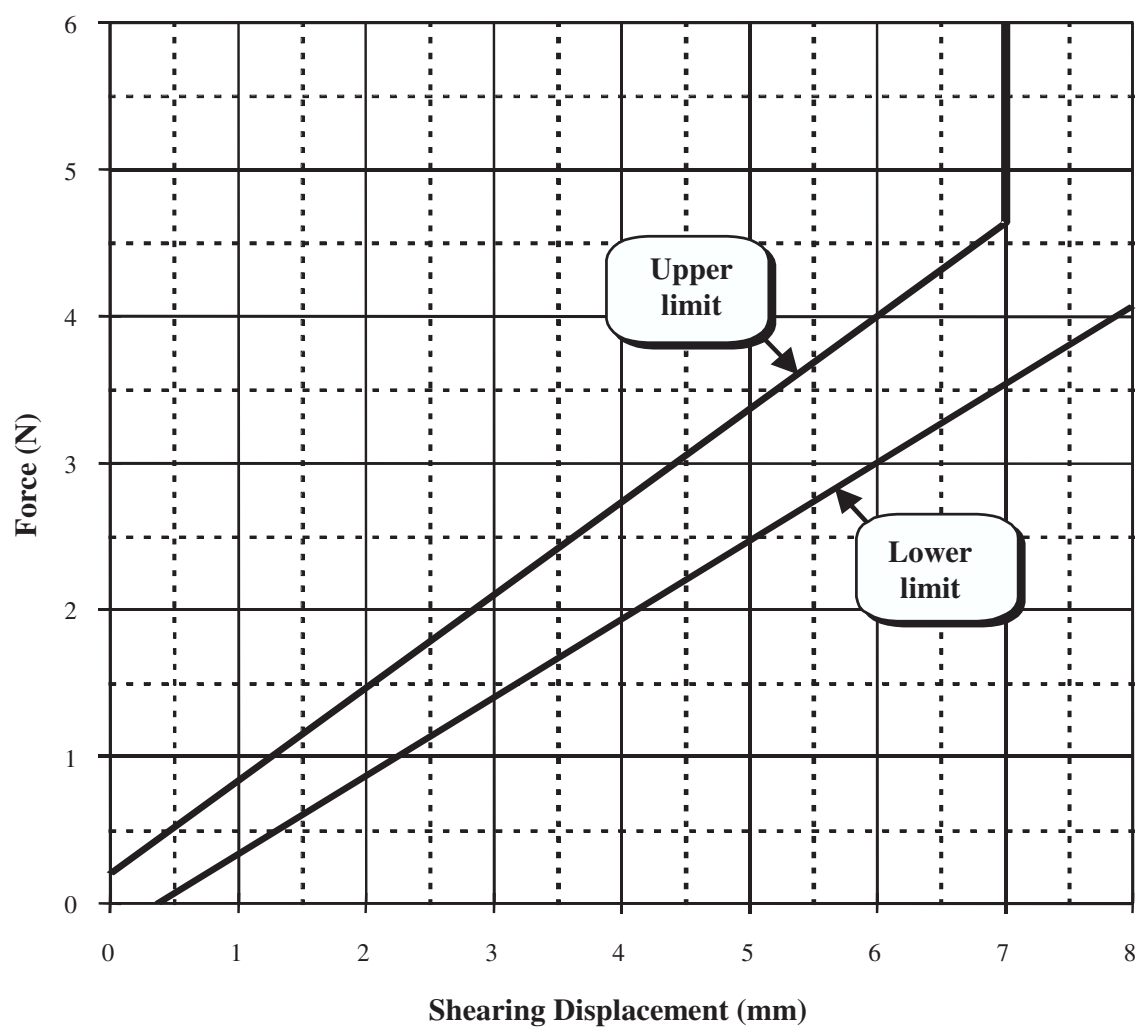
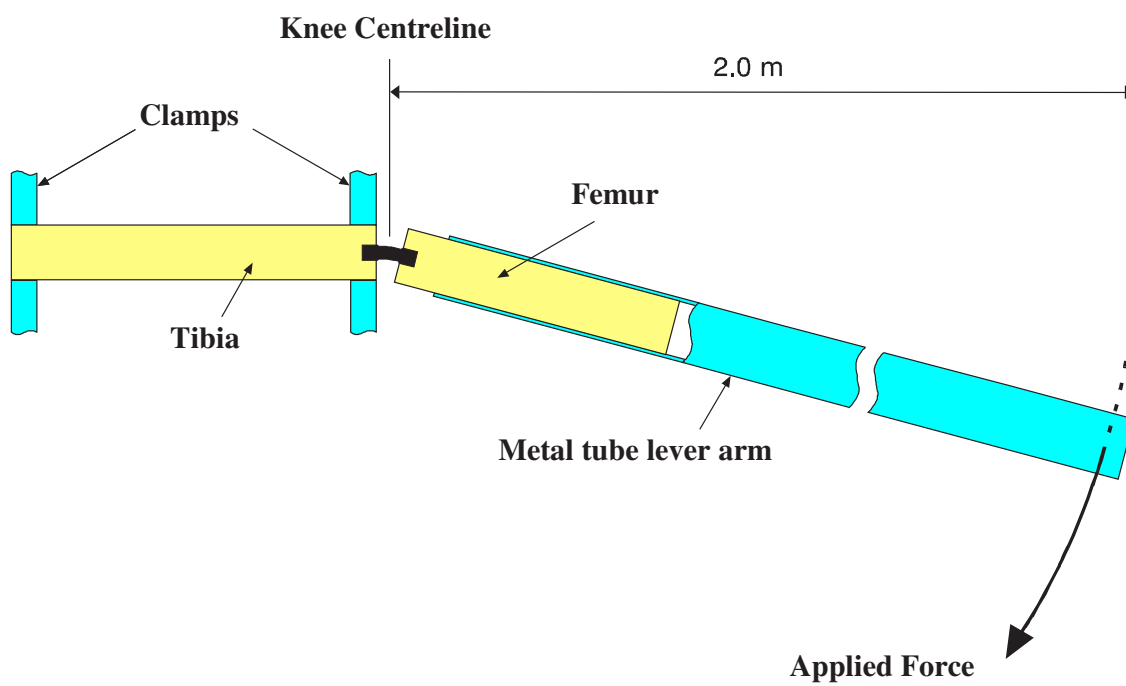


Figure 3

Test set-up for static legform impactor bending certification test

**Figure 4**

Test set-up for static legform impactor shearing certification test

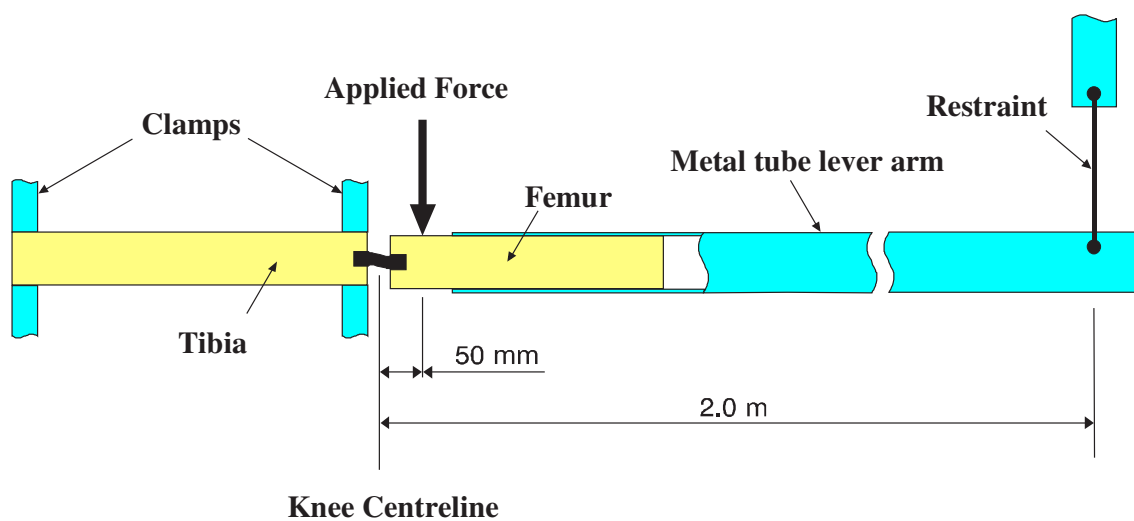


Figure 5a

Test set-up for dynamic legform impactor certification test
(side view top diagram, view from above bottom diagram)

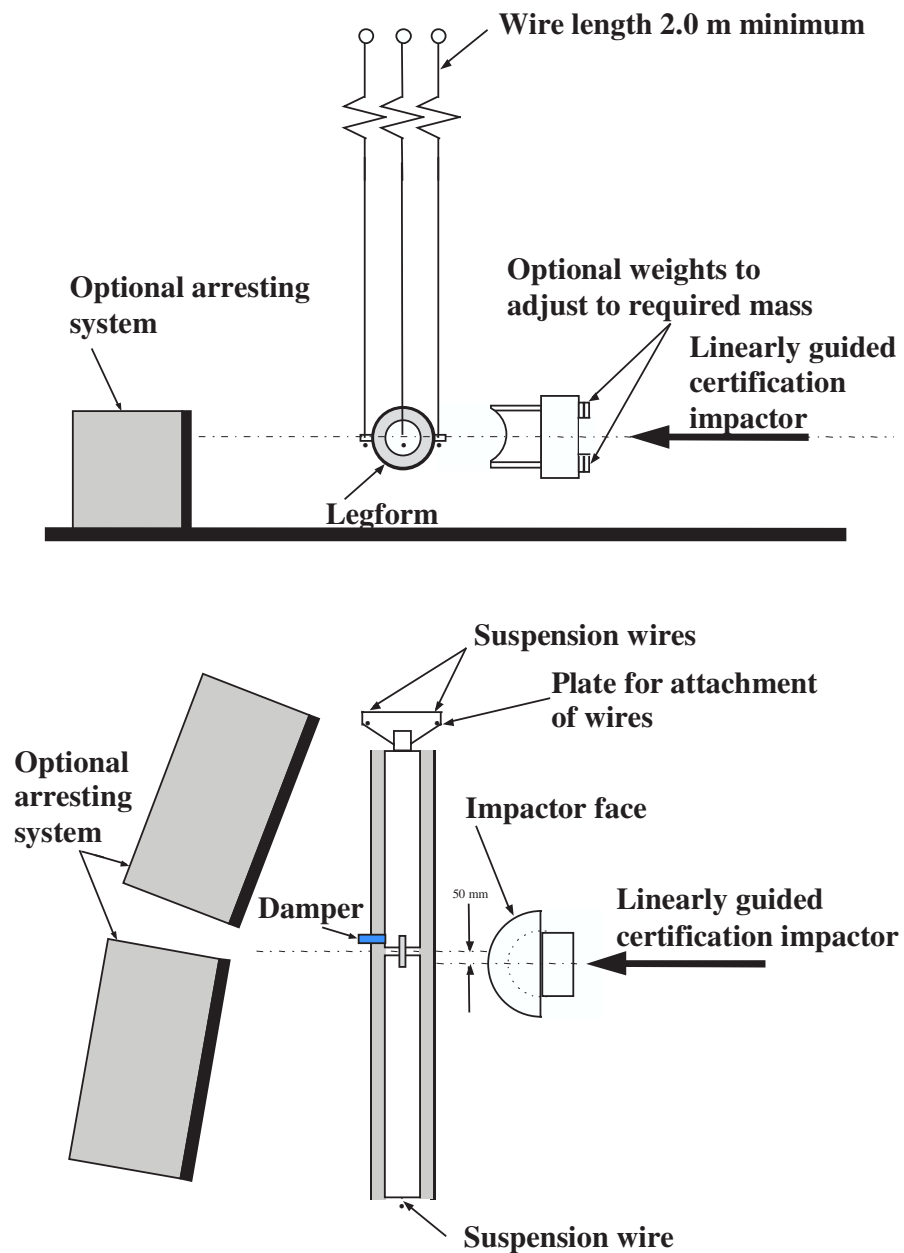
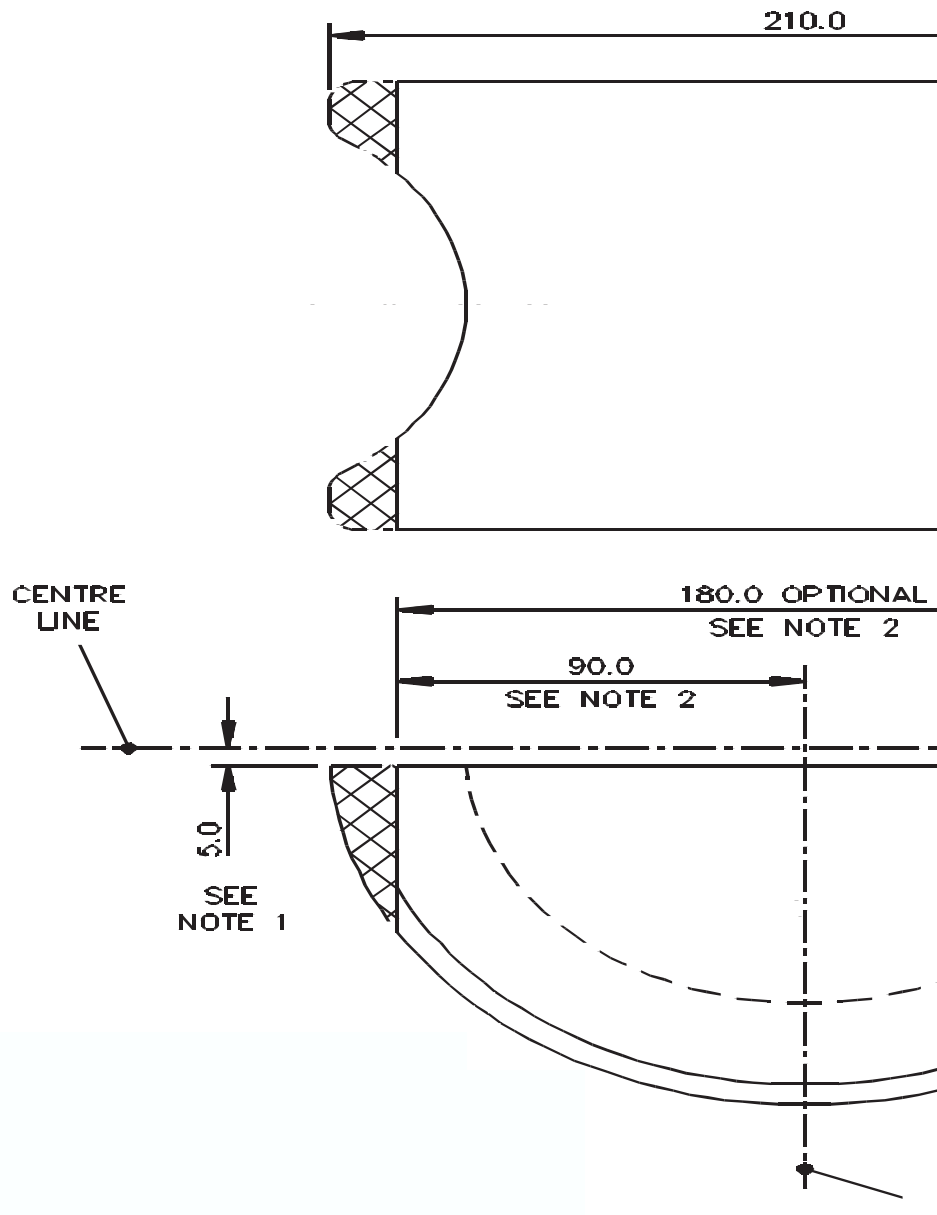


Figure 5b

Details of dynamic legform certification impactor face



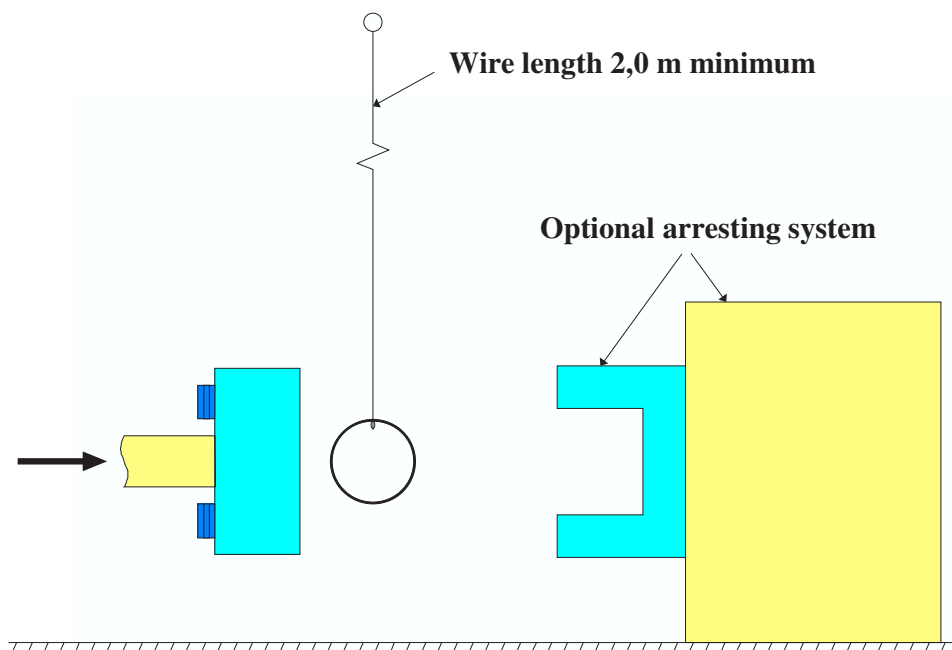
Notes:

1. Saddle may be made as a complete diameter and cut as shown to make two components.
2. The shaded areas may be removed to give the alternative form shown.
3. Tolerance on all dimensions is ± 1.0 mm.

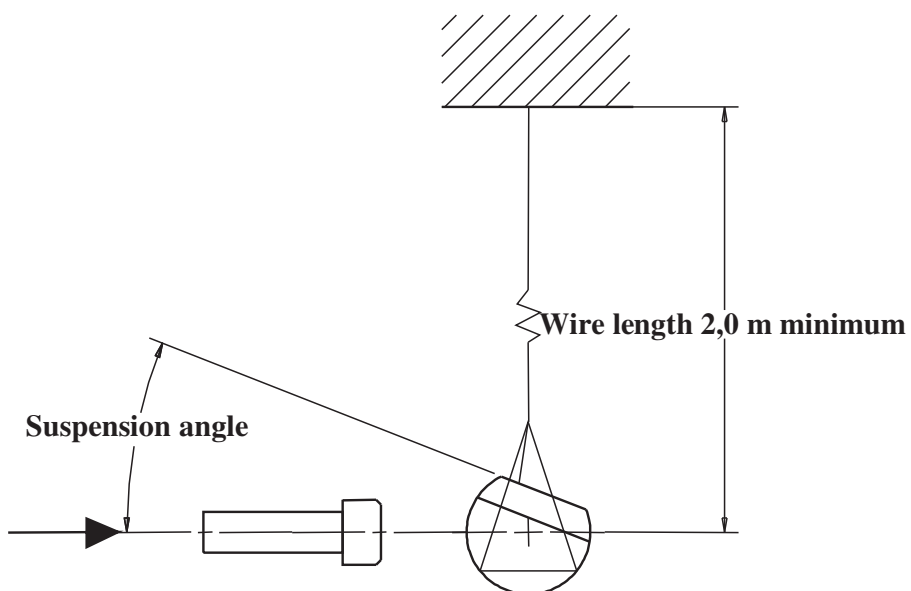
Material: Aluminium alloy

Figure 6

Test set-up for dynamic upper legform impactor certification test

**Figure 7**

Test set-up for dynamic headform impactor certification test



ANNEX IV

ACTIVE SAFETY SYSTEMS THAT COULD AVOID OR REDUCE PEDESTRIAN INJURIES IF PLACED ON CARS

1. Automatic windscreen wipers;
2. Pressure measuring sensors on tyres;
3. Electronic stability programmes (ESP);
4. Parking sensors adapted to hidden children behind;
5. Wide angled mirrors on all models (cyclists protection);
6. Enhanced vision systems;
7. Smart headlights, vary in intensity and/or directions;
8. Adaptive speed limiting devices on engines/maximum speed advisory devices (telematic speed adjust/recommendation);
9. Electronic pedestrian reflectors + vehicle receivers;
10. Car driving black boxes;
11. Automatic emergency call location devices;
12. Sensors which warn/slow down cars when pedestrian is in front (to be developed);
13. External airbags.